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BORE EVACUATOR VALVE TEST, CANNON 155MM HOWITZER, M126

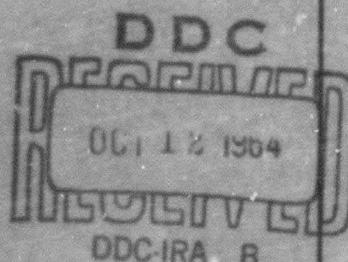
# TECHNICAL REPORT

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# BORE EVACUATOR VALVE TEST, CANNON 155mm HOWITZER, M126

## ABSTRACT

The limited life of Bore Evacuator Valve Assembly 8769384 during firing tests led to the authorization of a test program to find a valve assembly with a longer life. The cost of testing in the gun (155mm How. M126) made it economical to build a test apparatus which simulated the weapon. The test program was the basis for the incorporation of valve assembly 8769531 into the weapon system. A comparison of the strain level of the modification is presented. The life of the then current production valve assembly and the new production valve assembly under different charges is also given.

## Cross-Reference Data

Artillery  
Howitzers  
Cannon,155mm T255E3  
Cannon,155MM How M126  
Bore Evacuation  
Valves  
Bore Scavenging

## CONCLUSIONS AND RECOMMENDATIONS

Instrumented firing resulted in the choice of new valve assembly 8769531 as the assembly to be fatigue tested against the current production valve assembly 3769384. The former combination of components had the lowest strain level of all the assemblies tested.

During the fatigue test on valve assemblies 8769531 and 8769384 each had an average life of 6890 and 400 rounds, respectively, before crack initiation.

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## INTRODUCTION

During test firing of Cannon, 155mm Howitzer, M126, it was found that the life expectancy of the bore evacuator valve assemblies could be markedly increased. A design change was authorized as a product improvement measure. To obtain correlation with calculated stress data, limited tests were made on valves fitted with strain gages and fired in the cannon.

Since it was planned to test several design modifications, and the cost to fire the valves in the cannon was expected to be high, equipment was designed which would simulate conditions of pressure and stress of the valve body while using the facilities available at Watervliet Arsenal.

## OBJECTIVE

The object of this program was to test several valve modifications of bore evacuator valve design for Cannon, 155mm How., M126. The valve modification with the lowest stress level was to be test fired along with valve assembly 8769384 to determine relative life expectancy.

## MATERIAL AND APPARATUS

A 30 caliber M1903 Springfield bolt action rifle, firing grenade launcher ammunition, was used as the pressure generator for the test. An adapter (Figure 5) was made to secure the valve assembly in line with the muzzle end of the barrel, which allowed the charge to be fired directly into the valve. Another adapter (Figure 6) was made to give end pressure readings from the first adapter. The weapon was mounted in a specially built stand shown in Figures 1 and 2.

Figure 2 is a cross section drawing of the valve and adapter installed on the end of the rifle barrel. The length of the barrel was modified to obtain pressures similar to those the valve would experience when installed in the cannon.

Two oscilloscopes shown on Figure 3a were used to record the data, a Tektronix Type 535 scope with a 53-54C dual trace plug-in unit recorded pressures in conjunction with a Kistler Pressure Pickup Unit and a Tektronix Type 502 scope which took the strain readings. Figure 3b shows a closeup of the strain gage junction box, the compensating gages and a strain gaged valve body. A schematic wiring diagram, for both pressure and strain gage circuits, is shown in Figure 4.

The eleven valve assemblies compared with the current production valve assembly were made by combining the different modifications proposed. There were four different valves used in the assemblies. Figure 7 shows Valve 8769383, which was the then current production valve. The first modified Valve RDI-B7214, is shown in Figure 8. It was made by drilling a 3/8" diameter by 1/2" deep hole in the center of the top of the valve. Modification two, Valve RDI-B7215, shown in Figure 9, extended this hole to a depth of 5/8" and the third valve modification, Valve 8769529 (Figure 10), continued this hole to a depth of 1.8 inches. The then current production valve body, Body (Valve) 8769382, is shown in Figure 11. Body (Valve) RDI-B7212, Figure 12, was made by shortening Body (Valve) 8769382 by .272 inches. The other modified valve body, Body (Valve) 8769530, Figure 13, replaced the multiple holes with two long slots. Figure 14 shows Cap 8769381 used with valve bodies 8769382

and 8769530. Cap RDI-B7213, Figure 15, was used with Body (Valve) RDI-B7212.

The twelve valve assemblies are shown in Figures 16 - 27. Figures 16 - 19 are of Body (Valve) 8769382 with its Cap 8769381, and the four valves 8769381, RDI-B7214, RDI-7215 and 8769529 in this order. Figures 20, 21, 22 and 23 show Body (Valve) RDI-B7212 and its Cap RDI-B7213 with the various valves in the order above. Finally, Figures 24-27 show Body (Valve) 8769530 and Cap 8769381 with the four valves as listed above.

#### PROCEDURE

The .30 caliber Springfield rifle was modified to accomodate the bore evacuator valve adapter on its barrel. It was found necessary to insert an orifice in the adapter to raise the pressure and completely burn the propellant. As maximum pressure obtainable with the full barrel was too low, the length of the barrel was reduced. This raised pressure level permitted changes to the orifice size for the desired pressure, and/or strain.

With each change in orifice size, a series of rounds was fired using a valve assembly of the type shown in Figure 16, which had been fitted with strain gages and the strains were recorded. The strain gages were located on the valve body in the same position as those on the valve body which had been fired several rounds in a Cannon, 155mm Howitzer, M126 at Erie Proving Ground (see Figure 28). It was decided to change the orifice size until strains in the valve body on the test rig approached as nearly as possible the strains recorded from the valve assembly fired at the proving ground. The data from the proving ground is contained in the

memo dated 18 September 1963 titled "Firing Test to Determine Strains in the 155mm Howitzer T255E3, Bore Evacuator Valve During Firing", which is included in Appendix I.

After the optimum orifice size was determined, each of the modified assemblies was tested and the strain levels were compared with those recorded on Valve Assembly 8769384, Figure 16. Figure 29 shows the location of the strain gages on the short valve body, Body (Valve) RDI-B7212. The strain gage locations for the slotted valve body, Figure 13, are shown in Figure 30. Observation of the stress pattern on a valve body, shown in Figure 32, to which "Stress Coat", manufactured by the Magnaflux Corp., had been applied, led to the relocation of four strain gages on Body (Valve) 8769382. This can be seen by comparing Figures 28 and 31. The results of the test firing are given in Table I. Table II shows the "Reduction in Strain with Various Modifications." Valve Assembly 8769531, Figure 21, had the lowest strain readings and was chosen for use in the life test.

Seven (7) valve assemblies were used in the fatigue test. Four (4) were valve assembly 8769384. These were designated standard valves 1 through 4 or S.V. 1, etc. Three (3) valve assemblies 8769531 were designated modified valves 1 through 3, abbreviated M.V. 1, etc. Table 4 shows the average life obtained for the valve assemblies fired in the test program and in the cannon with different propellant charges. Table 3 gives the life of each of the valves fired in the Watervliet Arsenal test program.

During the life test, the valve assemblies were subjected to magnetic

particle inspection every 50 rounds. Figures 33 and 34 are standard valve 1 after 1000 rounds and modified valve 3 after 10,000 rounds, respectively. These pictures are taken under the conditions of magnetic particle inspection and the crack indications in the thread runout are shown by the arrows. Both valve assemblies were macroetched with a 50 per cent HCL solution for 30 to 45 minutes. Figure 35 shows 3 photos of modified valve 3. Figure 36 shows standard valve 1 and a Body (Valve) 8769530 cut and etched as above, after 1000 rounds of 100-115 per cent RMP at Erie Proving Ground.

Due to the long life experienced, an attempt was made to shorten this life by means of a cold test. It was felt that if the temperature of the valve body could be lowered to the vicinity of the ductile-brittle transition temperature, the fatigue life of the test pieces could be shortened. Although the cold test apparatus lowered the initial valve body temperature, the rate of fire to keep the temperature of the valve body at a low temperature was so slow that this procedure was unsatisfactory. This phase of the test was abandoned after 300 rounds, due to the problems of additional time to fire, shorter runs, and inaccessibility of the valve assembly to check the valve between rounds.

#### RESULTS AND DISCUSSION

The slotted valve body, Figure 13, reduced the stress concentration in the body area caused by the multiple hole configuration of the Body (Valve) 8769382, Figure 11. The more uniform stress distribution allowed the body to absorb more of the impact energy as the impact wave traveled to the fixed end; this also helped to reduce the strain at that end.

A well rounded, larger, radius in the thread runout did not reduce the strain level in this area. Therefore, the slotted body with the normal thread runout was chosen as the best possibility.

Table IV shows a comparison of the proof house life tests results with the firing data from Erie Proving Ground and Aberdeen Proving Ground. Since the proof house testing was based primarily on the strains registered on gages attached to the valve body and pressure was a secondary consideration, it seems reasonable that the increase in valve life obtained in the proof house must be reduced by a factor which allows for differences in pressures, volume of gases, and time of exposure. This is especially true as the stresses induced approach the endurance limit of the material. A slight decrease in stress would result in a substantial increase in life. The XM119E1 charge increased the initial pressure experienced by the valve to 120 per cent of the pressure generated by the 115 per cent RMP round. This great increase in pressure and consequently in stress decreases the life significantly. Another possibility, at present not fully investigated, is that one round in the 155mm M126 may result in more than one cycle of stress applied to the valve body.

#### ACKNOWLEDGEMENT

The authors wish to express their appreciation for the assistance given them by the Experimental Mechanics and Thermodynamics Lab of the Research Laboratories at Watervliet Arsenal in conducting the firing tests and obtaining the necessary data.

## APPENDIX

**DISPOSITION FORM**

(AR 340-15)

OFFICE SYMBOL OR FILE REFERENCE	SUBJECT
SWENV-RDR:rrl	FIRING TEST TO DETERMINE STRAINS IN THE 155MM HOW. T255E3 BORE EVACUATOR VALVES DURING FIRING
TO Industrial Engineering	FROM Mr. Ralph Lasselle
	DATE 18 Sept 63 CMT 1 Mr. Lasselle/mnf/5518
<p>1. Four valves were strain gaged at Watervliet Arsenal and taken to Erie Proving Ground for test. The valves were designated A thru D and are as follows:</p> <p><u>Valve A</u> (drawing number 8769384) is the valve currently in production. This valve had eight gages symmetrically spaced on the cylindrical portion of the body just above the conical valve seat. All gages were orientated to measure axial strain.</p> <p><u>Valve B</u> was similar to valve A in both geometry and gaging.</p> <p><u>Valve C</u> (drawing number WTV-C9640) had the same overall dimensions as valves A and B, but had two longitudinal slots cut in the side for passage of the propellant gases rather than the fourteen holes and the radius of the relief for the mounting threads had been increased to decrease the stress concentration factor. The valve also had a poppet which weighed 3 3/8 oz. compared to the 3 3/4 oz. for the ones in valves A and B. The gaging on this valve was the same as on valves A and B plus four additional gages mounted in the thread relief. All gages were orientated to measure axial strain.</p> <p><u>Valve D</u> was similar to valve C except that it had a cap weighing 3 5/8 oz. compared to the cap on valves A, B and C which weighed 5 7/16 oz. The gaging on valve D was the same as on valve C.</p> <p>2. The axis of the valves were at an angle of 30° to the axis of the gun tube providing only a partial seat on the side of the valve away from the gun tube. This caused a non-uniform strain in the thread relief when torqued down to the 75 ft. lbs. recommended. The strain as measured in the gages on valve C due to mounting torque were 200/<math>\mu</math> in/in compression in the gages adjacent to the partial seat and 500/<math>\mu</math> in/in tension in the gages opposite the previously mentioned gages (gages between the axes of the bore evacuator valve and gun tube).</p> <p>3. The strain gage data consisted of damped oscillations whose frequency ranged between 600 cps and 1,000 cps. There are two separate blocks of oscillation for each firing. The first one starts with initiation of weapon recoil and is probably due to setback forces loading the valve as a cantilever. The vibrations last for about 6 to 7 milliseconds and then are overridden by larger vibrations which occur at about the time the projectile passes the valve port. These vibrations are probably set up by the poppet action. These oscillations are damped out in approximately ten cycles. The strains reported are peak to peak vibrations with the first figures for the setback initiated vibrations and the second figures for the poppet initiated vibrations.</p>	

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BORE EVACUATOR VALVE FIRING TEST  
TABLE OF STRAIN GAGE READINGS

Gage Location	A (Body)	B (Body)	C (Body)	C (Seat)	D (Body)	D (Seat)
Between axes of valve and gun tube	0 - lost	0 - lost	200 - lost	lost - lost	200 - lost	200 - lost
	400 - lost	400 - lost	200 - lost	300 - lost	300 - lost	lost - lost
	400 - lost	400 - lost	300 - lost	lost - lost	300 - lost	300 - lost
	400 - 888	400 - 2000	300 - 960	0 - lost	lost - lost	lost - lost
	ten 90- 100% cycles at 1000 cps					
2	Opposite first gage listed (adjacent to partial bore evacuator valve seat)	200 - 640	lost - lost	100 - 960	300 - 3555 twelve 90- 100% cycles at 600 cps	200 - lost
		200 - 1600	300 - 710	200 - 1000	300 - 1600 six 90-100% cycles at 600 cps	lost - lost
		lost - lost	300 - lost	100 - lost	500 - 2000 nine 90- 100% cycles at 800 cps	300 - lost
		200 - lost	500 - lost	lost - lost	0 - lost	lost - lost

SUBJECT: FIRING TEST TO DETERMINE STRAINS IN THE 155MM HOW.  
T255E3 BORE EVACUATOR VALVES DURING FIRING

4. Conclusions and Recommendations:

Due to the urgency of the project there was not sufficient time to develop a strain gage technique to get a complete set of data on all the gages, nor could one valve be fired more than one round with the same gages which is necessary to obtain the effects of reducing poppet or cap weight. One strain reading in the thread relief of valve C gave a strain of  $3555 \mu$  in/in which corresponds to a simple stress of 105,000 psi. All gages in the area between the axes of the valve and gun tube were lost at the time the gases opened the poppet (as can be seen in the first line of the table), and so no peak dynamic strains were obtained at this point experiencing  $500 \mu$  in/in tension due to assembly torque. It should also be remembered that the measured strains are not necessarily the maximum strains in the piece because the strain gage is an averaging device being used in a stress concentration area and that there is no guarantee that the gage was placed at the point of maximum strain.

A check of the frequency associated with various modes of vibrations of the valve indicates that the strains measured which were at 600 to 1000 cps were probably due to the valve vibrating as a cantilever beam with a mass at the end even though the major excitation was an axial impulse. The longitudinal shock wave mode might have been excited, but because its natural frequency is in the order of 15kc, it could not have been picked up by the instrumentation used.

The data seems to indicate quite clearly that the strains caused by poppet action are several times larger than those caused by setback forces. If it is determined that the fatigue lives of these valves are not satisfactory, even when it is considered that for each round fired the valve experiences approximately ten major stress cycles, it is recommended that a test device be developed that will primarily simulate poppet action.



RALPH R. LASSELLE

TABLE I  
BORE EVACUATOR VALVE TEST FIRING DATA  
WITH OPTIMUM ORIFICE .1562" IN DIAMETER

ROUND	PRESSURE KSI	GAGE NO	STRAIN $\mu$ IN./IN.	GAGE NO	STRAIN $\mu$ IN./IN.	BODY VALVE	VALVE
153	10.	8	1500	2	1499	STD <sup>1</sup>	ST <sup>2</sup>
157	10.	8	1392	2	1564	STD	ST
160	10.	8	1294	2	1514	STD	ST
161	10.	8	1294	3	929	STD	ST
162	10.	8	1270	4	1515	STD	ST
163	10.5	8	1343	6	1344	STD	ST
164	10.	8	1343	7	684	STD	ST
165	10.	8	1171	7	489	STD	M1 <sup>3</sup>
166	10.	8	1146	6	1171	STD	M1
167	10.5	8	1303	4	1564	STD	M1
168	9.5	8	1320	3	732	STD	M1
169	--	8	1171	2	1367	STD	M1
170	11.5	8	1292	2	1463	STD	M1
171	10.	8	805	2	927	STD	M1
172	10.5	8	879	3	464	STD	M2 <sup>4</sup>
173	10.	8	879	4	976	STD	M2
174	10.	8	879	6	830	STD	M2
175	10.	8	879	7	244	STD	M2
176	9.5	8	488	7	196	STD	M3 <sup>5</sup>
177	10.	8	439	7	196	STD	M3
178	10.	8	488	6	487	STD	M3
179	10.	8	438	4	544	STD	M3
180	10.5	8	439	3	342	STD	M3
181	10.5	8	410	2	512	STD	M3
183	10.75	1	214	8	1045	MB1 <sup>6</sup>	ST
184	9.5	8	1522	1	192	MB1	ST
185	9.75	8	1423	2	1205	MB1	ST
186	9.5	8	1423	3	482	MB1	ST
187	10.5	8	1545	4	1260	MB1	ST
188	11.	8	1708	5	341	MB1	ST
189	10.5	8	1708	6	1783	MB1	ST
190	10.5	8	1611	7	684	MB1	ST
191	10.	8	1516	9	---	MB1	ST
192	10.	8	1439	9	3710	MB1	ST
193	10.	8	1805	10	4500 <sup>+</sup>	MB1	ST
194	10.	8	1689	11	4340	MB1	ST
195	10.	8	1196	11	2976	MB1	M1
196	10.	8	1408	9	3705	MB1	M1
197	10.5	8	1630	7	684	MB1	M1
198	10.5	8	976	6	889	MB1	M1

TABLE I (CONT.)

**BORE EVACUATOR VALVE TEST FIRING DATA  
WITH OPTIMUM ORIFICE .1562" IN DIAMETER**

ROUND	PRESSURE KSI	GAGE NO	STRAIN $\mu$ IN./IN.	GAGE NO	STRAIN $\mu$ IN./IN	BODY VALVE	VALVE
199	10.	8	1333	5	297	MB1	M1
200	11.5	8	1300	4	1250	MB1	M1
201	11.	8	1234	3	643	MB1	M1
202	10.	8	1333	2	1284	MB1	M1
203	10.	8	1300	1	198	MB1	M1
207	--	8	1043	1	174	MB1	M1
208	10.	8	843	1	88	MB1	M2
209	9.5	8	913	2	740	MB1	M2
210	10.5	8	804	3	326	MB1	M2
211	10.5	8	956	4	807	MB1	M2
212	10.5	8	914	5	174	MB1	M2
213	10.5	8	826	6	913	MB1	M2
214	10.	8	870	7	383	MB1	M2
215	9.25	8	739	9	1956	MB1	M2
216	11.25	8	956	11	2366	MB1	M2
217	10.	8	326	11	870	MB1	M3
218	9.75	8	326	9	913	MB1	M3
219	9.25	8	356	9	847	MB1	M3
220	11.	8	345	7	165	MB1	M3
221	10.	8	354	6	441	MB1	M3
222	9.75	8	322	5	66	MB1	M3
223	12.	8	418	4	315	MB1	M3
224	10.	8	444	3	152	MB1	M3
225	11.5	8	409	2	380	MB1	M3
226	8.75	8	261	1	---	MB1	M3
230	10.	8	662	2	805	MB2 <sup>7</sup>	ST
231	10.5	8	840	3	892	MB2	ST
232	8.	8	831	4	1273	MB2	ST
233	9.	8	817	5	846	MB2	ST
234	10.75	8	953	6	646	MB2	ST
235	10.	8	1021	7	768	MB2	ST
236	11.5	8	1134	9	---	MB2	ST
237	10.5	8	1158	9	---	MB2	ST
238	10.5	8	1169	9	5910	MB2	ST
239	10.	8	844	9	4775	MB2	ST
240	10.	8	977	10	4555	MB2	ST
241	9.	8	933	11	5540	MB2	ST
242	10.5	8	965	12	4345	MB2	ST
248	--	8	910	12	4505	MB2	M1
249	9.	8	998	12	4395	MB2	M1

TABLE I (CONT.)

**BORE EVACUATOR VALVE TEST FIRING DATA  
WITH OPTIMUM ORIFICE .1562" IN DIAMETER**

ROUND	PRESSURE KSI	GAGE NO	STRAIN $\mu$ IN./IN.	GAGE NO	STRAIN $\mu$ IN./IN	BODY VALVE	VALVE
250	9.5	8	910	11	5420	MB2	M1
251	10.	8	955	10	3904	MB2	M1
252	10.5	8	933	9	4830	MB2	M1
253	9.5	8	907	7	653	MB2	M1
254	10.5	8	1066	6	609	MB2	M1
255	10.	8	953	5	764	MB2	M1
256	10.	8	908	4	1031	MB2	M1
257	12.25	8	726	3	733	MB2	M1
258	10.	8	1134	2	653	MB2	M1
259	11.	8	1089	1	702	MB2	M1
262	9.	8	636	12	2750	MB2	M2
263	11.25	8	635	11	4180	MB2	M2
264	9.5	8	636	10	3190	MB2	M2
265	8.75	8	704	9	4180	MB2	M2
266	11.5	8	746	7	638	MB2	M2
267	11.	8	704	6	440	MB2	M2
268	9.5	8	590	5	618	MB2	M2
269	9.5	8	703	4	880	MB2	M2
270	10.	8	749	3	704	MB2	M2
271	9.5	8	704	2	616	MB2	M2
272	10.75	8	703	1	440	MB2	M2
273	9.5	8	322	12	1650	MB2	M3
274	10.	8	242	12	1363	MB2	M3
275	8.75	8	356	11	--	MB2	M3
276	9.75	8	322	11	2284	MB2	M3
277	9.	8	230	9	1584	MB2	M3
278	10.	8	345	10	1716	MB2	M3
279	9.5	8	276	7	264	MB2	M3
280	9.5	8	311	6	264	MB2	M3
281	10.	8	368	5	418	MB2	M3
282	9.5	8	299	4	451	MB2	M3
283	10.	8	368	3	377	MB2	M3
284	8.	8	291	2	244	MB2	M3
285	9.25	8	356	1	189	MB2	M3
286	--	3	1862	6	225	S3 <sup>8</sup>	ST
287	9.5	3	1842	6	270	S3	ST
288	10.	3	1910	4	1012	S3	ST
289	9.	3	1793	2	2709	S3	ST
290	10.25	-	----	2	2812	S3	ST
291	9.25	3	1931	2	2655	S3	ST

TABLE I (CONT.)

**BORE EVACUATOR VALVE TEST FIRING DATA  
WITH OPTIMUM ORIFICE .1562" IN DIAMETER**

ROUND	PRESSURE KSI	GAGE NO	STRAIN $\mu$ IN./IN.	GAGE NO	STRAIN $\mu$ IN./IN.	BODY	VALVE
292	11.	3	2022	7	3330	S3	ST
293	9.25	3	1840	8	2790	S3	ST
294	9.75	3	604	8	1229	S3	M3
295	9.5	3	644	8	1137	S3	M3
296	9.75	3	605	7	1023	S3	M3
297	10.75	3	651	2	1136	S3	M3
298	10.25	3	628	2	1146	S3	M3
299	9.5	3	697	7	1318	S3	M3
300	8.25	3	604	8	1000	S3	M3
301	--	3	628	-	--	S3	M3
302	9.75	3	651	-	--	S3	M3
303	10.25	3	1161	-	--	S3	M2
304	11.	3	1674	-	--	S3	M1
305	10.	3	1720	-	--	S3	ST
308	10.5	3	1862	-	--	S3	M1
309	10.5	3	1884	-	--	S3	M1
310	11.	3	1242	-	--	S3	M2
311	11.5	3	621	-	--	S3	M3

1 BODY (VALVE) B8769382, FIGURE 11; STRAIN GAGE LOCATIONS, FIGURE 28.

2 STANDARD PRODUCTION VALVE AT TIME OF TEST, FIGURE 7.

3 FIRST MODIFIED VALVE 1/2" DEEP HOLE FIGURE 8.

4 MODIFIED VALVE TWO 5/8" DEEP HOLE, FIGURE 9.

5 ADOPTED MODIFICATION FIGURE 10.

6 SHORTENED BODY (VALVE) RDI-B7212, FIGURE 12; STRAIN GAGE LOCATIONS, FIGURE 29.

7 ADOPTED MODIFIED BODY (VALVE) B8769530, FIGURE 13; STRAIN GAGE LOCATIONS, FIGURE 30.

8 BODY (VALVE) B8769382, FIGURE 11, STRAIN GAGE LOCATIONS, FIGURE 31.

TABLE II  
REDUCTION IN STRAIN  
WITH VARIOUS  
VALVE COMPONENTS

BODY VALVE*	GAGE NO	STRAIN $\mu$ IN./IN.			% OF ST VALVE			
		ST <sup>2</sup>	M1 <sup>3</sup>	M2 <sup>4</sup>	M3 <sup>5</sup>	M1	M2	M3
STD <sup>1</sup>	2	1536	1275	925	480	83.0	60.3	31.2
	3	930	770	440	325	82.9	47.4	35.0
	4	1515	1490	975	545	98.5	64.4	36.9
	6	1280	1170	830	480	91.5	64.8	37.5
	7	680	460	240	190	67.7	35.4	28.0
	8	1300	1185	855	445	91.2	68.0	34.2
MB1 <sup>6</sup>	1	205	195	88	---			
	2	1230	1280	780	330	104.0	63.4	26.8
	3	510	590	310	160	115.5	52.6	27.1
	4	1200	1090	760	260	91.8	63.4	21.7
	5	310	290	170	65	93.6	59.9	21.0
	6	1700	845	870	445	49.7	51.1	26.2
	7	650	650	385	145	100.0	59.3	22.3
	8	1560	1250	900	350	80.2	57.7	22.4
	9	3700	3700	2125	875	100.0	57.5	23.7
	11	4300	2950	2100	875	68.6	48.8	20.3
MB2 <sup>7</sup>	1	610	630	410	200	103.0	67.2	32.8
	2	805	655	650	305	81.4	80.7	37.9
	3	850	600	700	375	70.6	82.4	44.1
	4	1590	1035	925	480	65.0	58.2	30.2
	5	890	765	650	365	85.0	73.0	41.0
	6	600	590	400	420	98.5	65.7	70.0
	7	765	690	555	275	90.2	72.5	35.9
	8	940	990	685	335	105.2	72.9	35.6
	9	5200	4600	4750	1740	88.5	91.4	33.5
	10	4550	3900	3350	1740	85.7	73.6	38.2
	11	6150	5700	3725	2350	93.5	60.6	38.2
	12	4150	4850	3040	1550	117.0	73.4	37.4

\* KEY AT BOTTOM OF TABLE I APPLIES

TABLE III  
NUMBER OF ROUNDS ON VALVE ASSEMBLIES  
AT PROOF HOUSE OF WATERVLIET ARSENAL

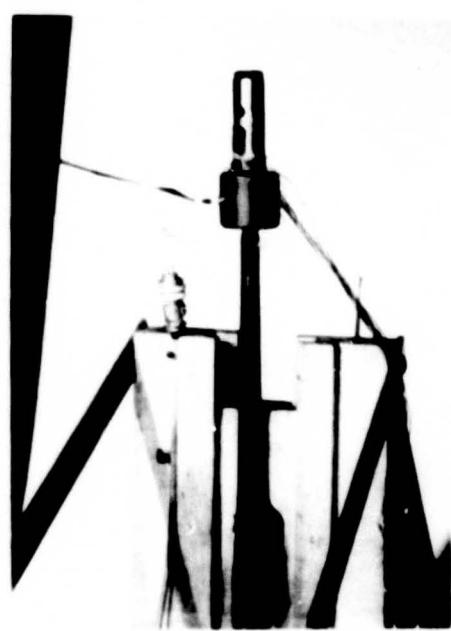
ASSEMBLY NUMBER	TEST DESIGNATION	ROUNDS UNTIL FIRST INDICATION	TOTAL NUMBER OF ROUNDS
B8769384	S.V. 1	455	1000
B8769384	S.V. 2	501	1300
B8769384	S.V. 3	393	1000
B8769384	S.V. 4	145	1000
B8769531	M.V. 1	8051	10000
B8769531	M.V. 2	----	5000
B8769531	M.V. 3	5711	10000

TABLE IV  
AVERAGE NUMBER OF ROUNDS  
ON VALVE ASSEMBLIES  
FROM DIFFERENT CHARGES

LOCATION OF FIRING	PROOF HOUSE	EPC		
		TEST APPARATUS	100%+ 115%RMP	100%RMP
PROPELLANT CHARGE				XMI19E1
VALVE ASSEMBLY				
B8769531	6890	1000	----	230
B8769384	400	----	280	100



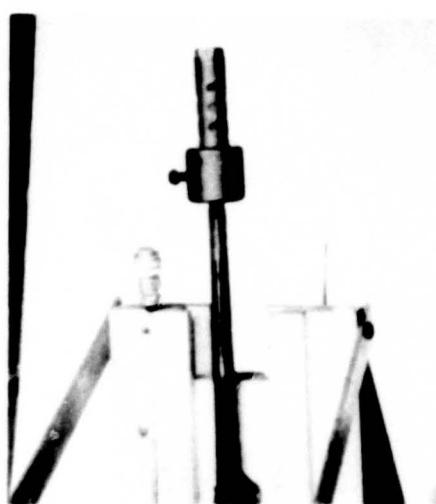
A



B



C

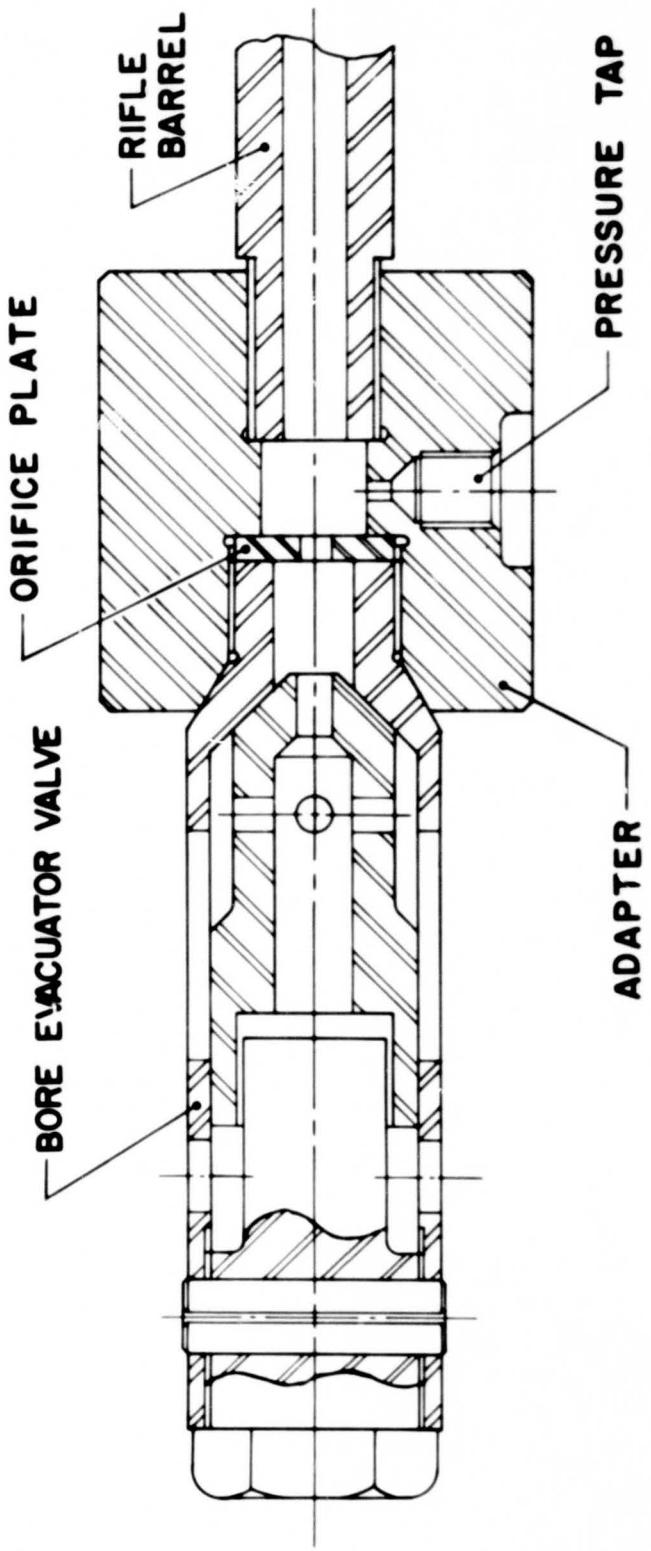


D

Figure 1: Test Apparatus

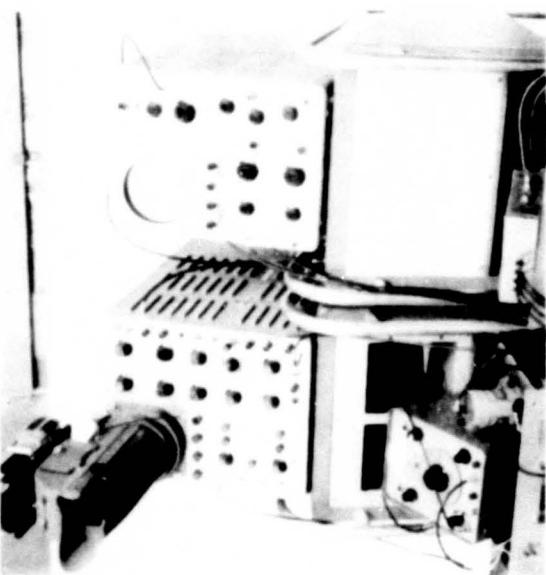
- A. General View of Test Stand.
- B. Close-up of strain-gaged valve assembly and side pressure pick-up.
- C. End pressure adapter and pressure pick-up in place of valve.
- D. Valve installed for fatigue test.

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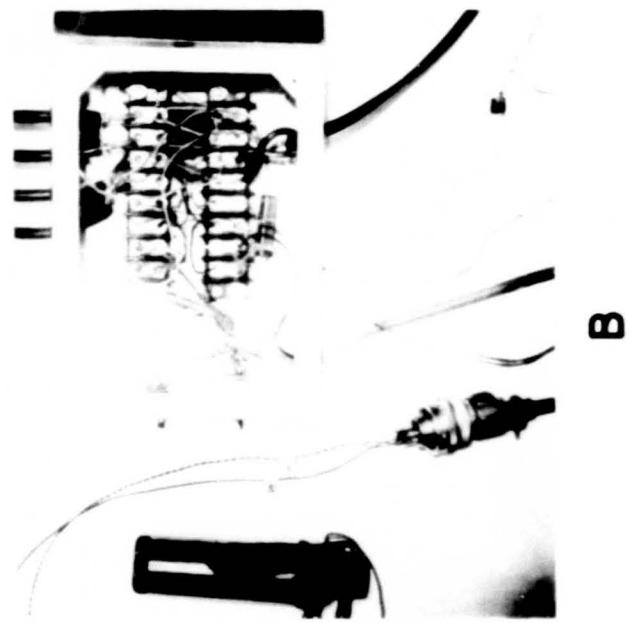


REVISIONS		DATE APPROVED		DEPT OF THE ARMY	
BY	DESCRIPTION	BY	DATE	WATERVLIET ARSENAL	WATERVLIET, N. Y.
				RDI-B7231	
				CODE	SHEET OF

FIGURE 2



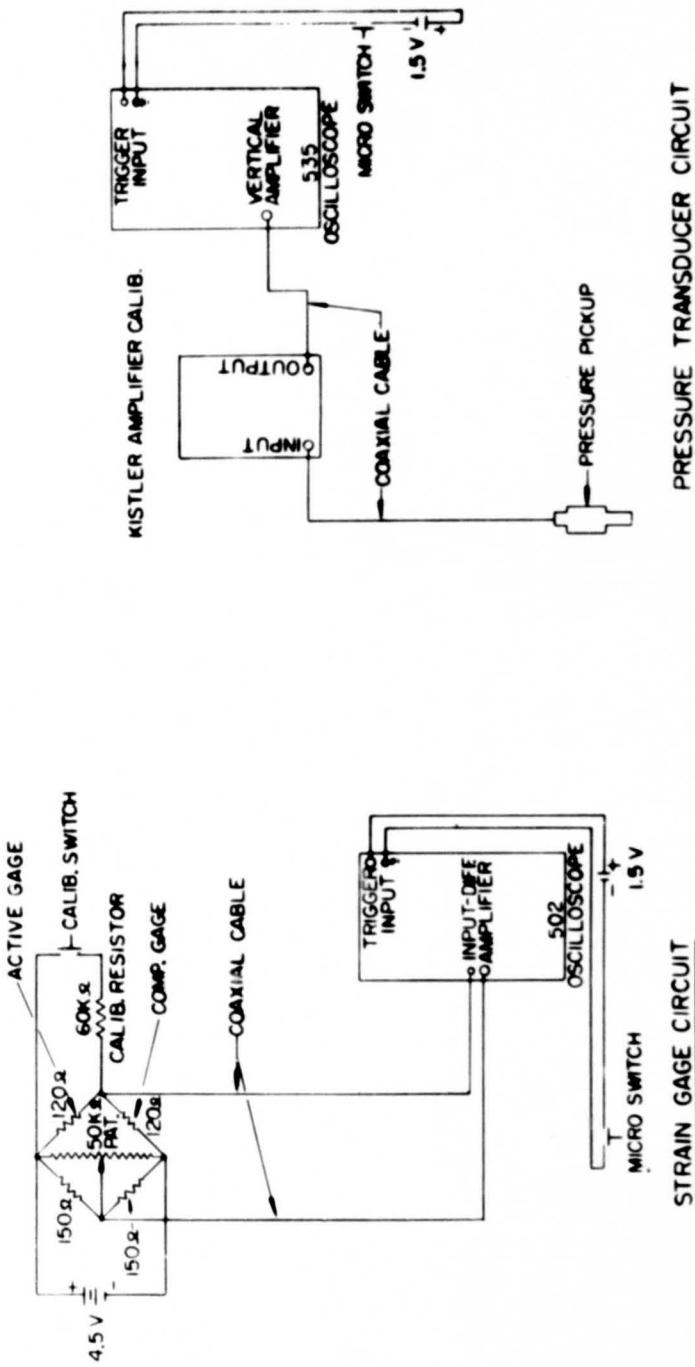
To the left Figure 3A  
Two oscilloscopes used to record data.  
Oscilloscope on left records pressure.  
Kistler calibration unit is shown below  
scope. Scope on right records strains.  
The strain gage calibration junction  
box is shown below scope.



To the right Figure 3B  
Close-up showing equipment  
used in Strain Gage Circuit.

Figure 3: Test Instruments

REVISIONS	
REV:	
DESCRIPTION:	
DATE:	
APPRO:	



		RDI-B7246		DEPT OF THE ARMY WATERVLIET ARSENAL WATERVLIET, N.Y.	
		WIRING DIAGRAMS (VALVE TESTS)		RDI-B7246	
NAME	GRADE	PHYSICAL PROPERTIES	ORIGINAL DATE OF DRAWINGS	REVIS	REVIS
		1.6 mm 1.6 mm	UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES TOLERANCES ON INCHES FRACTIONS ARE IN THOUSANDS OF AN INCH	1/2	1/2
DATE ENCL'D IN THERING RECORDS	USED ON	TEST AND FINAL ASMT	REVIS	REVIS	REVIS
TEST AND	APPLICATION	QTY REQD	REVIS	REVIS	REVIS
NOT	APPLY PART NO	AS SPECIFIED	REVIS	REVIS	REVIS
DO	AS SPECIFIED	DATES	REVIS	REVIS	REVIS

FIGURE 4

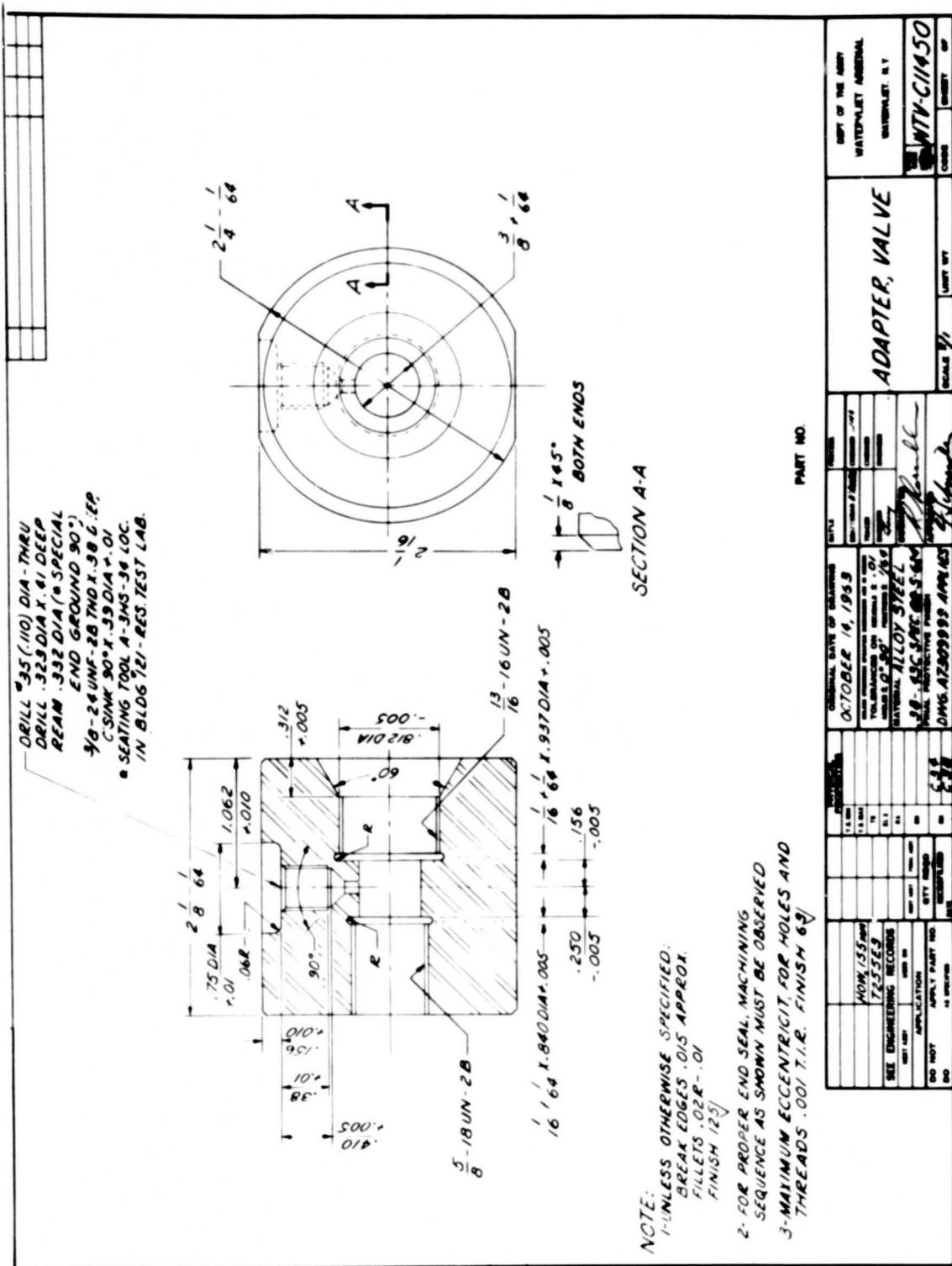


FIGURE 5

FIGURE 8

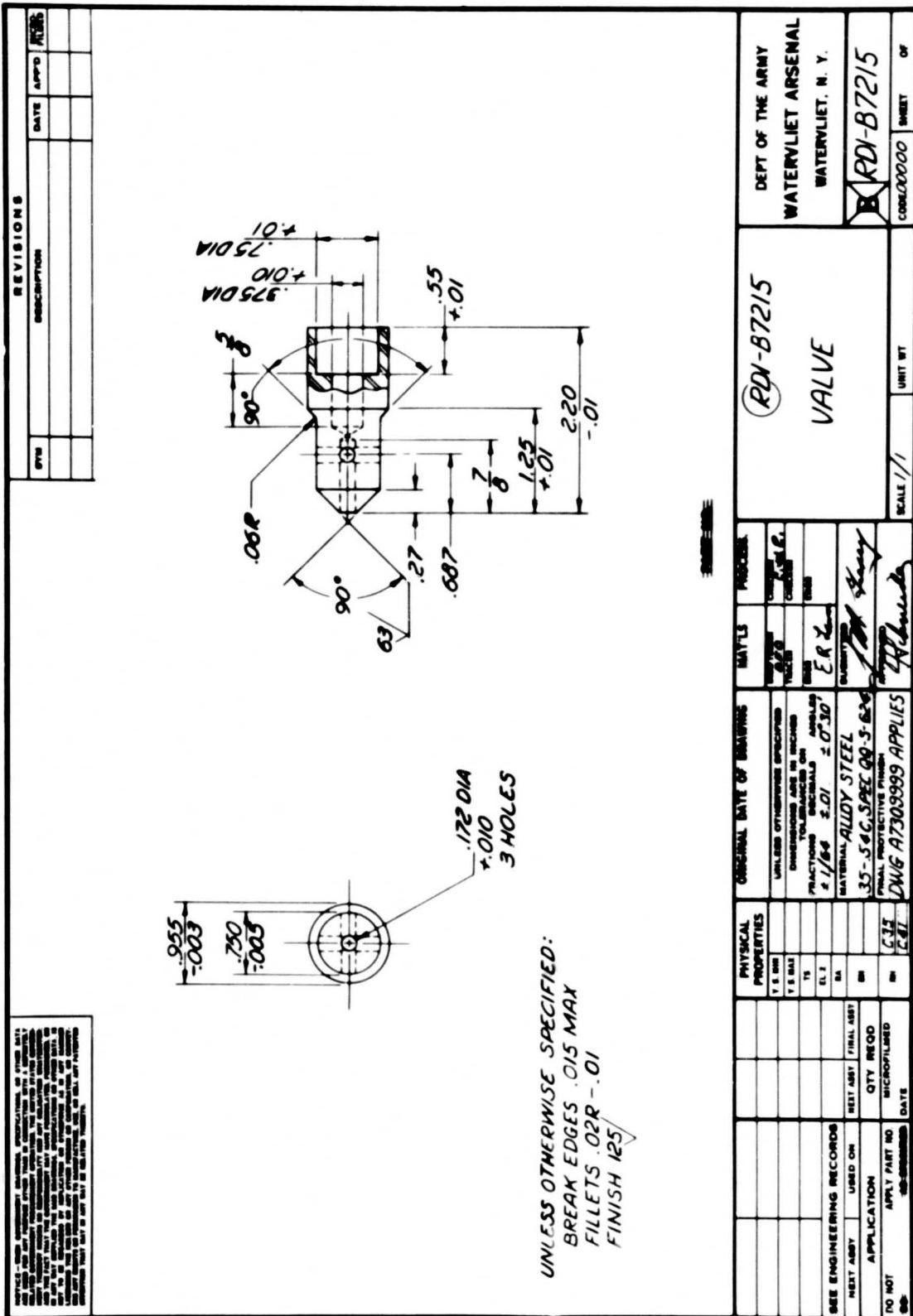


FIGURE 9

<small>NOTICE—WHEN GOVERNMENT DRAWINGS, SPECIFICATIONS, OR OTHER DATA ARE USED FOR ANY PURPOSE OTHER THAN IN CONNECTION WITH A DEFINITELY RELATED GOVERNMENT PROCUREMENT OPERATION, THE UNITED STATES GOVERNMENT THEREBY INCURS NO RESPONSIBILITY NOR ANY OBLIGATION WHATSOEVER AND THE FACT THAT THE GOVERNMENT MAY HAVE FORMULATED, FURNISHED, OR IN ANY WAY SUPPLIED THE SAID DRAWINGS, SPECIFICATIONS OR OTHER DATA IS NOT TO BE REGARDED BY IMPLICATION OR OTHERWISE AS IN ANY MANNER LICENSING THE HOLDER OR ANY OTHER PERSON OR CORPORATION OR CONVEYING ANY RIGHTS OR PERMISSION TO MANUFACTURE, USE OR SELL ANY PATENTED INVENTION THAT MAY IN ANY WAY BE RELATED THERETO.</small>															
<b>PHYSICAL PROPERTIES</b> VP TS EL 2 RA DN RH		<b>DO NOT USE</b> <b>APPLY PART NO.</b> <b>SEE E.O. 13104</b> <b>SEE ENGINEERING RECORDS</b> <b>8769531 HOW. 153 MM</b> <b>M126</b> <b>C35</b> <b>C41</b>	<b>REVISIONS</b> <table border="1"> <thead> <tr> <th>SYM</th> <th>DESCRIPTION</th> <th>DATE</th> <th>APPROVAL</th> </tr> </thead> <tbody> <tr> <td>NEW</td> <td>SEE E.O. 13104</td> <td></td> <td></td> </tr> <tr> <td>A</td> <td>SEE E.O. 13117</td> <td>1-16-69</td> <td></td> </tr> </tbody> </table>	SYM	DESCRIPTION	DATE	APPROVAL	NEW	SEE E.O. 13104			A	SEE E.O. 13117	1-16-69	
SYM	DESCRIPTION	DATE	APPROVAL												
NEW	SEE E.O. 13104														
A	SEE E.O. 13117	1-16-69													
<p><b>NOTE:</b>  <b>UNLESS OTHERWISE SPECIFIED:</b>  <b>BREAK EDGES .015 MAX</b>  <b>FILLETS .02R-.01</b>  <b>FINISH 125/</b></p>															
ORDNANCE PART NO. <b>8769529</b>															
<small>UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES TOLERANCES ON DECIMALS <math>\pm .01</math> FRACTIONS <math>\pm 1/16</math> ANGLES <math>\pm 5^\circ</math> MATERIAL ALLOY STEEL .35-. SPEC. QQ-S-616 HEAT TREATMENT SEE PHYSICAL PROPS. FINAL PROTECTIVE FINISH DWG A7309999 APPLIES</small>	<small>ORIGINAL DATE OF DRAWING DEC. 31 1968 DRAFTSMAN CHECKED <i>H. C. H.</i> TRACER CHECKED TELETYPE SUBMITTED <i>H. C. H.</i></small>	<b>VALVE</b> <i>H. C. H.</i>	<b>ORDNANCE CORPS DEPT OF THE ARMY WATERVLIET ARSENAL</b>												
<small>APPROVED BY CHIEF OF THE CHIEF OF ORDNANCE <i>R. L. Anatolian</i> ORD CORPS</small>		SCALE $1/4$ UNIT WT CODE 00000	<b>A</b> <b>8769529</b> SHEET OF												

FIGURE 10

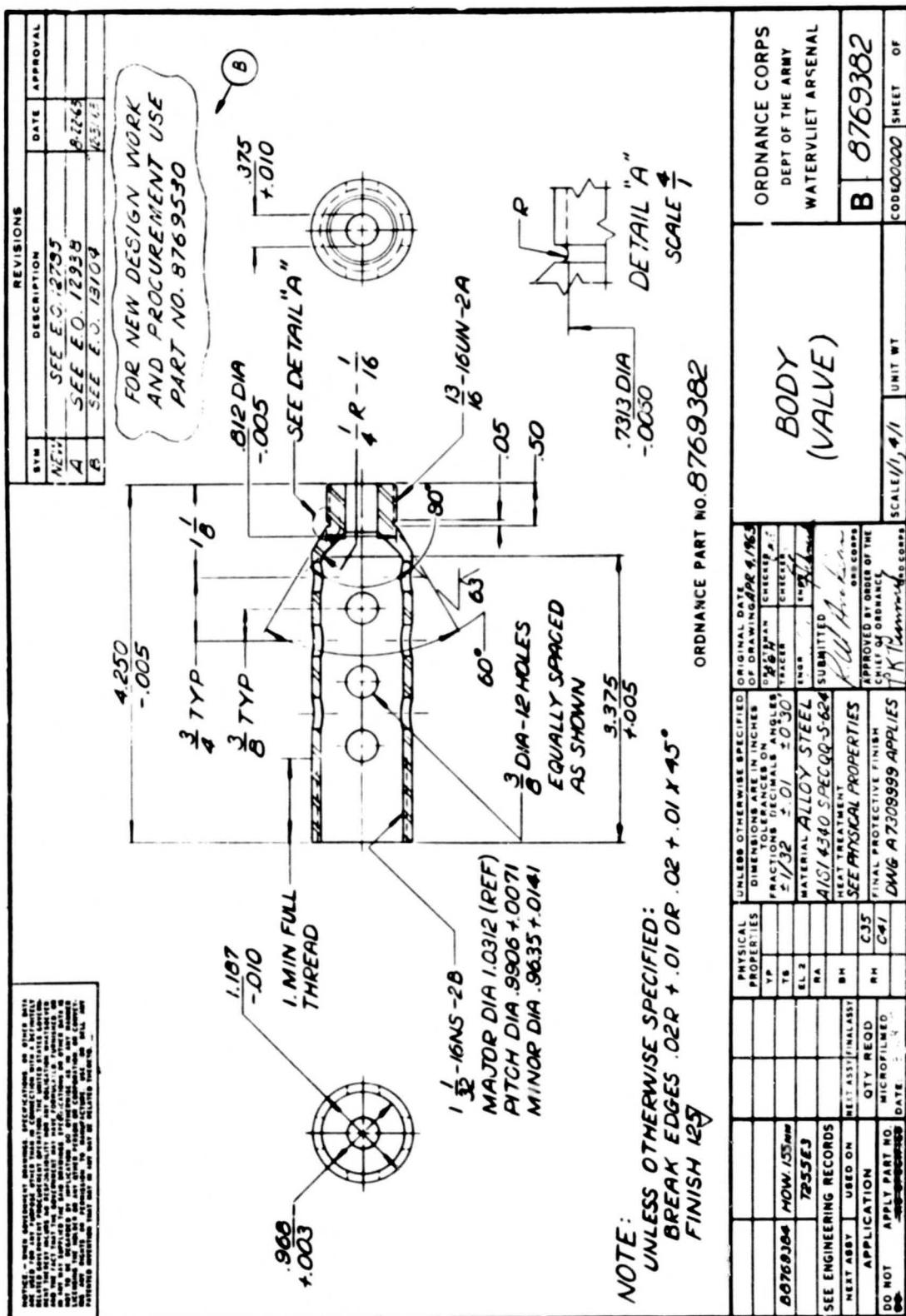


FIGURE 11

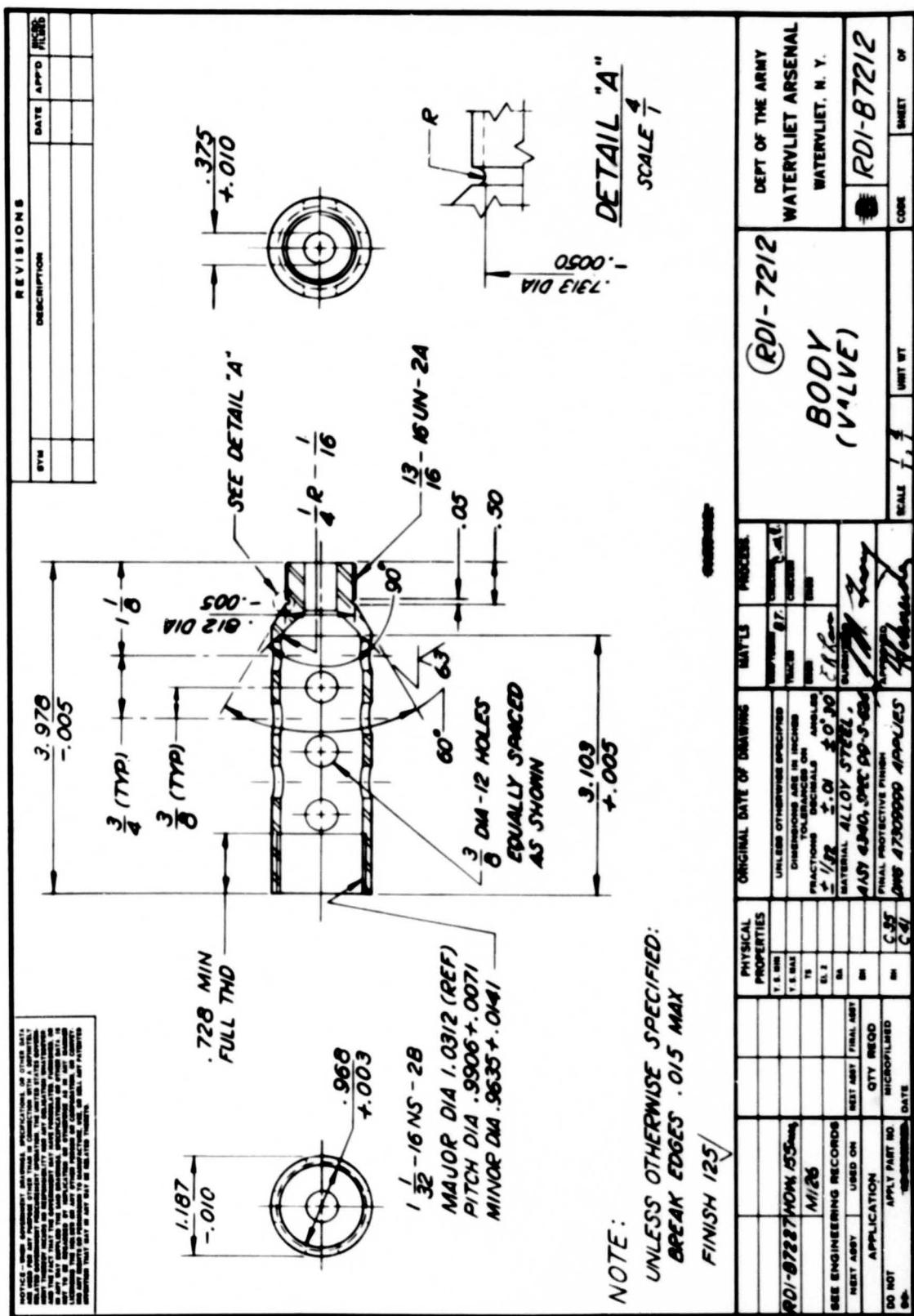


FIGURE 12

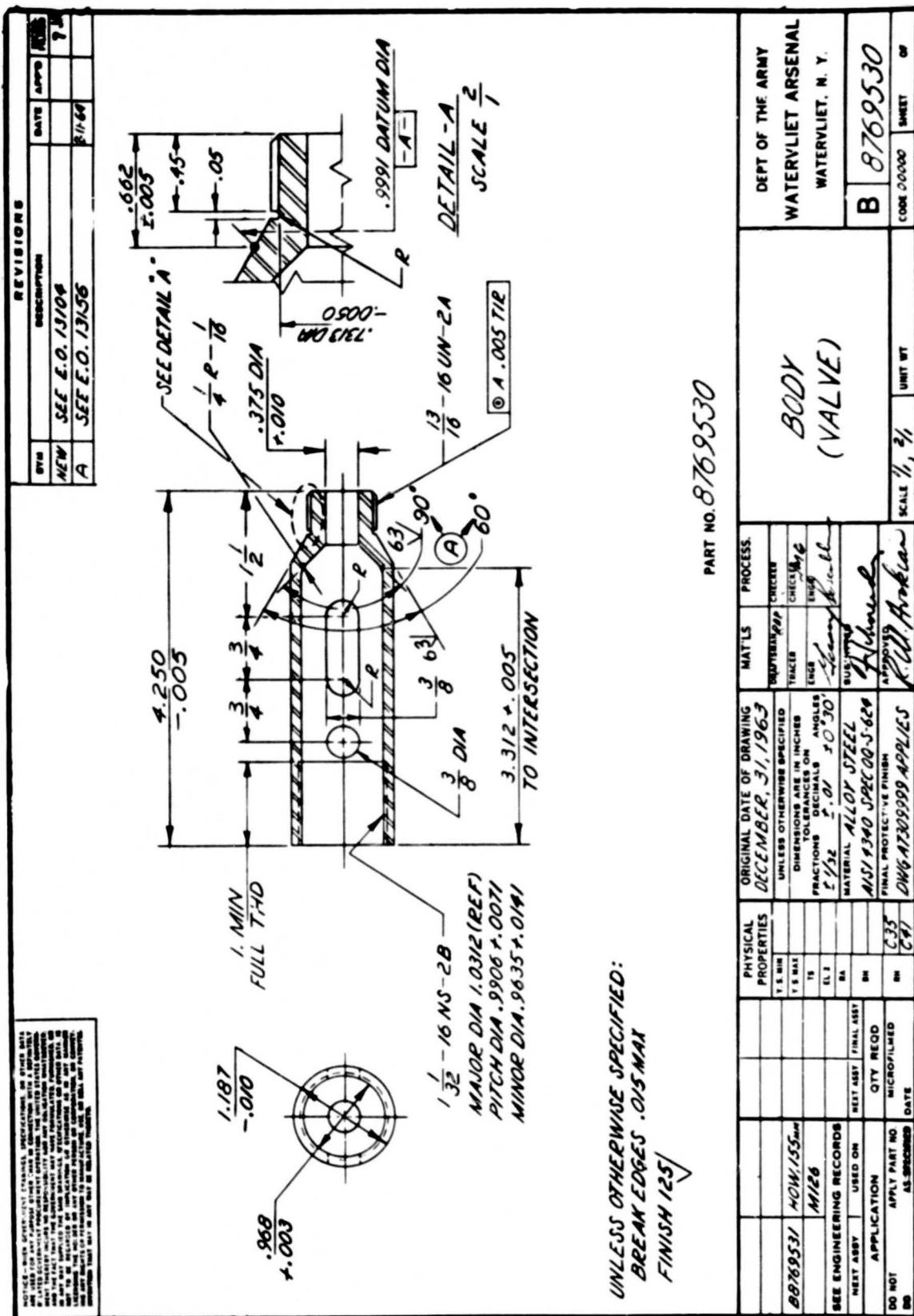
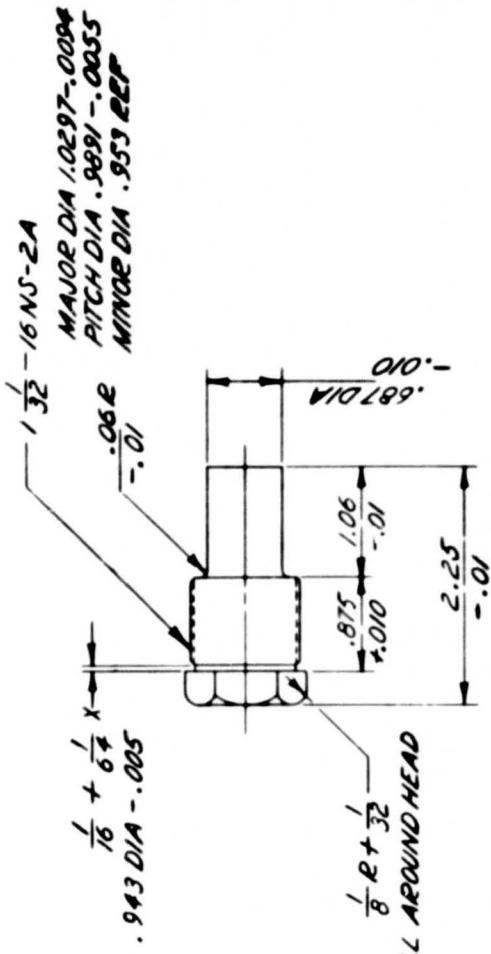


FIGURE 13

REVISIONS		DESCRIPTION	DATE	APPROVAL
REV				
NEW	<u>SEE E.O. 12705</u>			



UNLESS OTHERWISE SPECIFIED:  
BREAK EDGES .02 &+. 01 &+. 01 X 45°  
FILLETS .03 &+: A

FIRMSH 125

		UNLESS OTHERWISE SPECIFIED		ORIGINAL DATE APR 1943	ORDNANCE CORPS	
		DIMENSIONS ARE IN INCHES		DRAFTED BY S. C. TRACER	DEPT OF THE ARMY	
		TOLERANCES ON FRACTIONS DECIMALS ANGLES + OR - .010		CHECKED BY S. C. TRACER	WATERVLIET ARSENAL	
00769384		MATERIAL-ALLOY STEEL-.35C		SUPERVISOR S. C. TRACER		
T255E3		SPEC QQ-S-624		SUBMITTED S. C. TRACER		
SEE ENGINEERING RECORDS		HEAT TREATMENT		APPROVED BY ORDER OF THE CHIEF OF ORDNANCE PLANT MANAGER OF CORPS		
NEXT ABBY	USED ON	TEST ASSEMBLIES	BH			
APPLICATION	QTY REQD	BH	C35			
DO NOT	APPLY PART NO	MICROFILED	C48	SCALE 1/1	UNIT WT	CODE 00000
						SHEET OF

*PLUG*

B 0769384

FIGURE 14

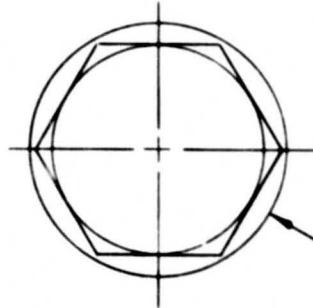
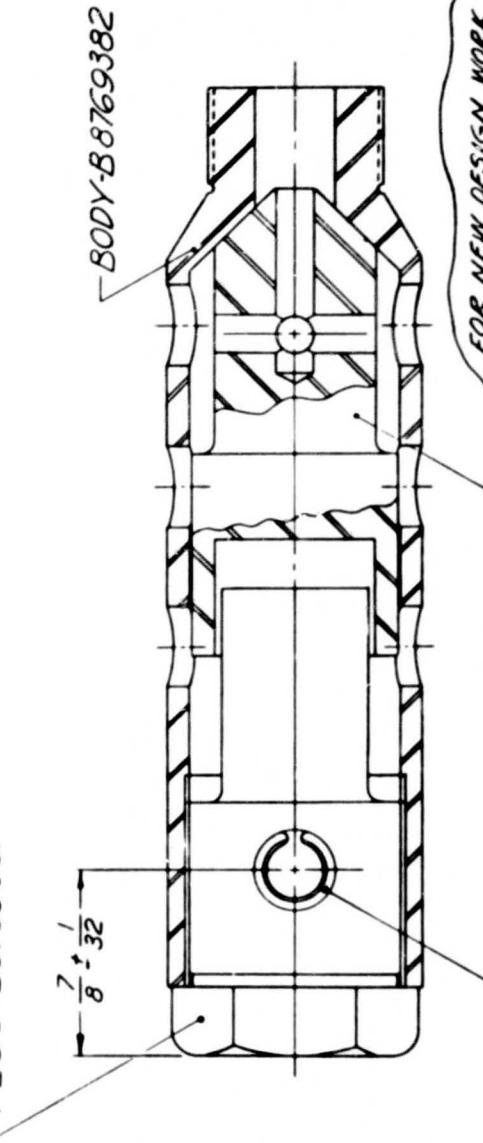


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REVISIONS

	DESCRIPTION	DATE APPROVED
NEW	SEE E.O. 12785	3/8/63
A	SEE E.O. 12967	
B	SEE E.O. 13104	12/3/63

PLUG-B8769381



APPLY PART NO.

PIN, SPRING-MS16562-172  
(.375 DIA+.005 HOLE DRILL  
THROUGH AT ASSEMBLY)

VALVE-B8769383

FOR LIST OF PARTS, SEE ENGINEERING PARTS LIST 8769384

ORDNANCE PART NO.8769384

FOR NEW DESIGN WORK  
AND PROCUREMENT  
USE PART NO. 8769381

ORDNANCE CORPS DEPT OF THE ARMY WATERVLIET ARSENAL		CODE 00000 SHEET <i>1</i> OF
<i>WATERVLIET ARSENAL</i>		
<i>VALVE ASSEMBLY</i>		<b>B</b> 8769384
SUBMITTED <i>J.W. [Signature]</i>		SCALE 2/1
APPROVED BY ORDER OF THE CHIEF OF ORDNANCE <i>J.W. [Signature]</i>		UNIT WT
SEE ENGINEERING RECORDS		FINAL PROTECTIVE FINISH
NEXT ASSY USED ON	NEXT ASSY FINAL ASSY	HEAT TREATMENT
APPLICATION	QTY REQ'D	MAINTENANCE
DO	MICROFINISHED AS SPECIFIED	DO

FIGURE 16

REVISIONS	DATE APPROVED	APPROVING OFFICER

PIN, SPRING MS/6562/172

VALVE RDI-B7214

PIN, SPRING MS/6562/172

PLUG-B8769381

BODY, VALVE-B8769382

RDI - 7216		DEPT OF THE ARMY WATERVILLE ARSENAL WATERVILLE, N. Y.
VALVE ASSEMBLY		RDI-B7216
SCALE //	UNIT OF	COMMITTEE
		REMARKS
SEE ENGINEERING RECORDS		
NEXT ARMY	USED ON	FINAL ARMY
APPLICATION	QTY REQD	INCHES/PLACES
DO NOT	APPLY PART NO	AS SPECIFIED
DO		

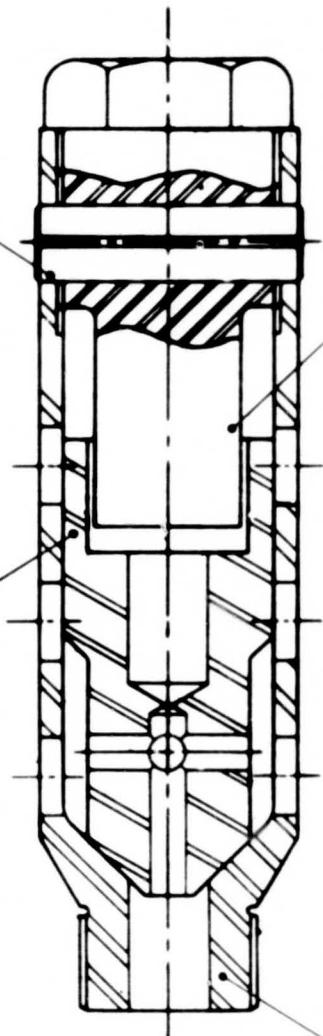
FIGURE 17

REVISIONS		DATE APPROVED	REVISION NUMBER
REV NO.	DESCRIPTION		



VALVE RD1-B 72/5

P/N, SPRING MS/6562/22



BODY, VALVE-B 88769382

PLUG-B 8769381

SEE ENGINEERING RECORDS		PHYSICAL PROPERTIES		CHEMICAL STATE OF MATTER		MANUFACTURER		DEPT OF THE ARMY	
ITEM NO.	DESCRIPTION	WEIGHT	TEMPERATURE	TYPE	COMPOSITION	ITEM NO.	DESCRIPTION	ITEM NO.	DESCRIPTION
1	VALVE	1.5 lbs	100° F	IRON	IRON	2	SPRING	3	WATERVLIET ARSENAL
2	SPRING	1.5 lbs	100° F	IRON	IRON	4	SCREW	5	WATERVLIET, N. Y.
3	SCREW	1.5 lbs	100° F	IRON	IRON	6	SCREW	7	RD1-B 72/7
4	SCREW	1.5 lbs	100° F	IRON	IRON	8	SCREW	9	RD1-B 72/7
5	SCREW	1.5 lbs	100° F	IRON	IRON	10	SCREW	11	SCREW
6	SCREW	1.5 lbs	100° F	IRON	IRON	12	SCREW	13	SCREW
7	SCREW	1.5 lbs	100° F	IRON	IRON	14	SCREW	15	SCREW
8	SCREW	1.5 lbs	100° F	IRON	IRON	16	SCREW	17	SCREW
9	SCREW	1.5 lbs	100° F	IRON	IRON	18	SCREW	19	SCREW
10	SCREW	1.5 lbs	100° F	IRON	IRON	20	SCREW	21	SCREW
11	SCREW	1.5 lbs	100° F	IRON	IRON	22	SCREW	23	SCREW
12	SCREW	1.5 lbs	100° F	IRON	IRON	24	SCREW	25	SCREW
13	SCREW	1.5 lbs	100° F	IRON	IRON	26	SCREW	27	SCREW
14	SCREW	1.5 lbs	100° F	IRON	IRON	28	SCREW	29	SCREW
15	SCREW	1.5 lbs	100° F	IRON	IRON	30	SCREW	31	SCREW
16	SCREW	1.5 lbs	100° F	IRON	IRON	32	SCREW	33	SCREW
17	SCREW	1.5 lbs	100° F	IRON	IRON	34	SCREW	35	SCREW
18	SCREW	1.5 lbs	100° F	IRON	IRON	36	SCREW	37	SCREW
19	SCREW	1.5 lbs	100° F	IRON	IRON	38	SCREW	39	SCREW
20	SCREW	1.5 lbs	100° F	IRON	IRON	40	SCREW	41	SCREW
21	SCREW	1.5 lbs	100° F	IRON	IRON	42	SCREW	43	SCREW
22	SCREW	1.5 lbs	100° F	IRON	IRON	44	SCREW	45	SCREW
23	SCREW	1.5 lbs	100° F	IRON	IRON	46	SCREW	47	SCREW
24	SCREW	1.5 lbs	100° F	IRON	IRON	48	SCREW	49	SCREW
25	SCREW	1.5 lbs	100° F	IRON	IRON	50	SCREW	51	SCREW
26	SCREW	1.5 lbs	100° F	IRON	IRON	52	SCREW	53	SCREW
27	SCREW	1.5 lbs	100° F	IRON	IRON	54	SCREW	55	SCREW
28	SCREW	1.5 lbs	100° F	IRON	IRON	56	SCREW	57	SCREW
29	SCREW	1.5 lbs	100° F	IRON	IRON	58	SCREW	59	SCREW
30	SCREW	1.5 lbs	100° F	IRON	IRON	60	SCREW	61	SCREW
31	SCREW	1.5 lbs	100° F	IRON	IRON	62	SCREW	63	SCREW
32	SCREW	1.5 lbs	100° F	IRON	IRON	64	SCREW	65	SCREW
33	SCREW	1.5 lbs	100° F	IRON	IRON	66	SCREW	67	SCREW
34	SCREW	1.5 lbs	100° F	IRON	IRON	68	SCREW	69	SCREW
35	SCREW	1.5 lbs	100° F	IRON	IRON	70	SCREW	71	SCREW
36	SCREW	1.5 lbs	100° F	IRON	IRON	72	SCREW	73	SCREW
37	SCREW	1.5 lbs	100° F	IRON	IRON	74	SCREW	75	SCREW
38	SCREW	1.5 lbs	100° F	IRON	IRON	76	SCREW	77	SCREW
39	SCREW	1.5 lbs	100° F	IRON	IRON	78	SCREW	79	SCREW
40	SCREW	1.5 lbs	100° F	IRON	IRON	80	SCREW	81	SCREW
41	SCREW	1.5 lbs	100° F	IRON	IRON	82	SCREW	83	SCREW
42	SCREW	1.5 lbs	100° F	IRON	IRON	84	SCREW	85	SCREW
43	SCREW	1.5 lbs	100° F	IRON	IRON	86	SCREW	87	SCREW
44	SCREW	1.5 lbs	100° F	IRON	IRON	88	SCREW	89	SCREW
45	SCREW	1.5 lbs	100° F	IRON	IRON	90	SCREW	91	SCREW
46	SCREW	1.5 lbs	100° F	IRON	IRON	92	SCREW	93	SCREW
47	SCREW	1.5 lbs	100° F	IRON	IRON	94	SCREW	95	SCREW
48	SCREW	1.5 lbs	100° F	IRON	IRON	96	SCREW	97	SCREW
49	SCREW	1.5 lbs	100° F	IRON	IRON	98	SCREW	99	SCREW
50	SCREW	1.5 lbs	100° F	IRON	IRON	100	SCREW	101	SCREW

FIGURE 18

REVISIONS		DESCRIPTION	DATE	APD	REC'D. FILE'D.
REV.	IN				

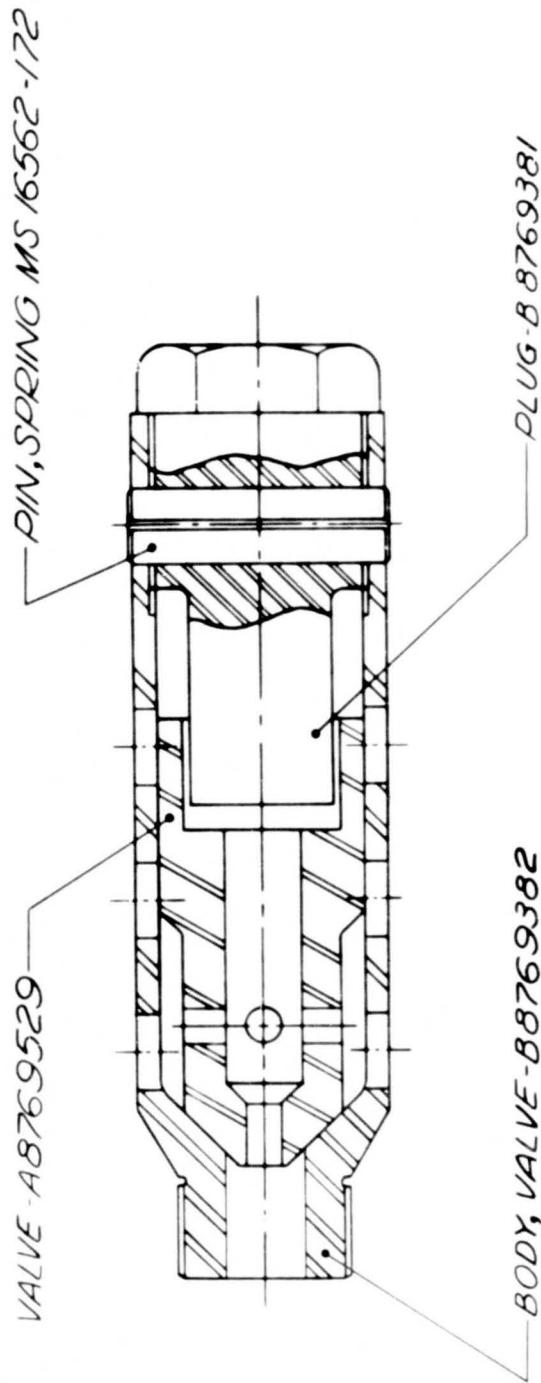
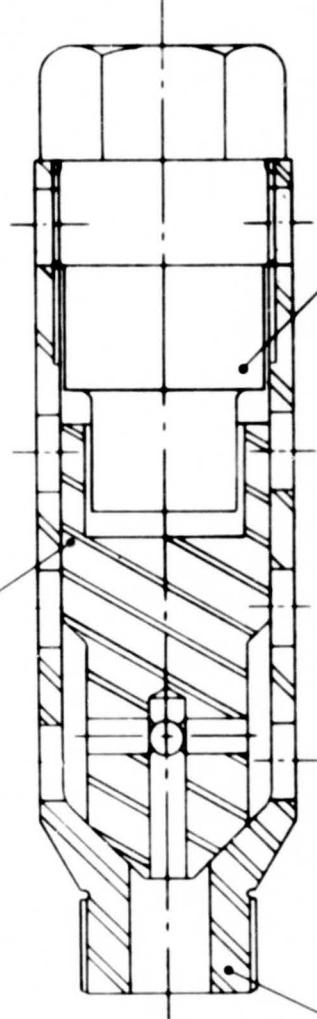


FIGURE 19

REVISIONS		DESCRIPTION	DATE APPROVED	APPROVING OFFICER
S/N				

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VALVE-B8769383



BODY, VALVE RDI-B722/2

PLUG RDI-B722/3

PHYSICAL PROPERTIES		GENERAL DATA OF PARTS	DATA	PROCESS	DEPT OF THE ARMY
Y.S. T.S. T.R. E.I. M.	Y.S. T.S. T.R. E.I. M.	UNLESS OTHERWISE SPECIFIED MATERIALS TO BE USED ON FRCTIONAL SURFACES	1000 1000 1000 1000 1000	BC C C C C	WATERVILLE ARSENAL WATERVILLE, N.Y.
SEE ENGINEERING RECORDS		MATERIAL			RDI-B7222
NEXT ACTV	USED ON	TEST ACTV (FINAL ACT)			VALVE ASSEMBLY
APPLICATION	QTY NEEDED	FINAL PROTECTIVE FINISH			RDI-B7222
DO NOT DO AS SPECIFIED	INCHES/INCHES				CODE SHEET OR

FIGURE 20

REVISIONS		DATE	APFD	WHD FILED
REV.	DESCRIPTION			

**VALVE RDI-B7214**

**PLUG RDI-B7213**

**BODY, VALVE RDI-B7212**

PHYSICAL PROPERTIES		ORIGINAL DATE OF DRAWING	MATERIAL	PROCESS
7 1/2 in. 7 1/2 in.		RDI-7223 UNLESS OTHERWISE SPECIFIED	STRUCTURAL STEEL TRACER FRICTION JACOB'S ANGLED E.P.C.	CHAMFERED CHECKED ENDS E.P.C.
1/8 in. 1/8 in.		1/8 in. 1/8 in.	FRACTIONAL DIMENSIONS IN INCHES INCHES JACOB'S ANGLED	
SEE ENGINEERING RECORDS				
NEXT ARMY USED ON		FINAL ARMY		
APPLICATION		QTY 1000 UNCHANGED		
DO NOT USE		APPLY PART NO AS SPECIFIED DATE		

DEPT OF THE ARMY WATERVLIET ARSENAL WATERVLIET, N.Y.		CODE	SUBJECT
RDI-B7223 VALVE ASSEMBLY			

FIGURE 21

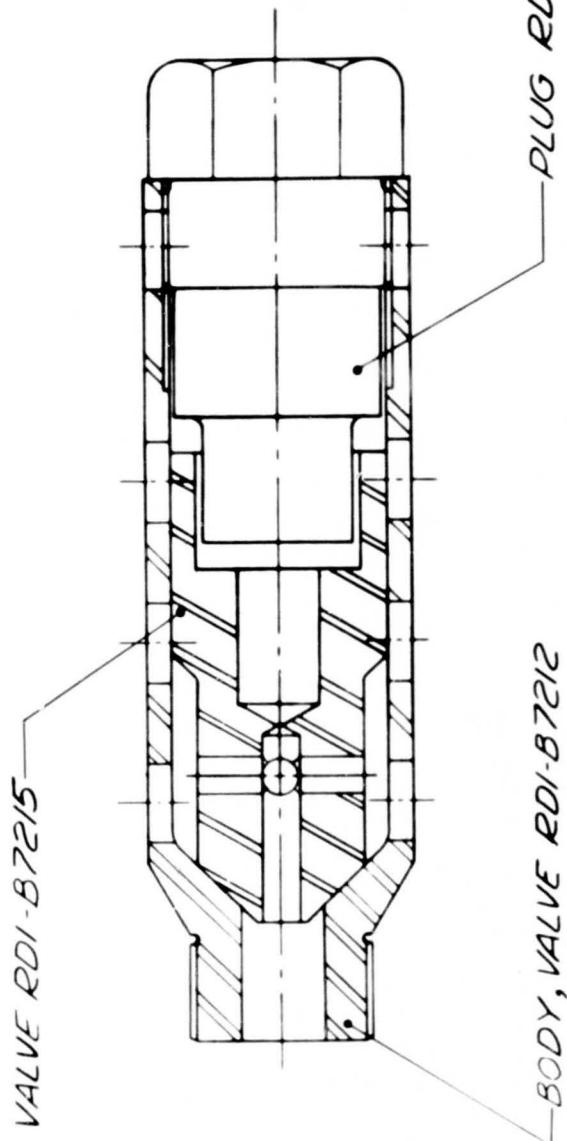


FIGURE 22

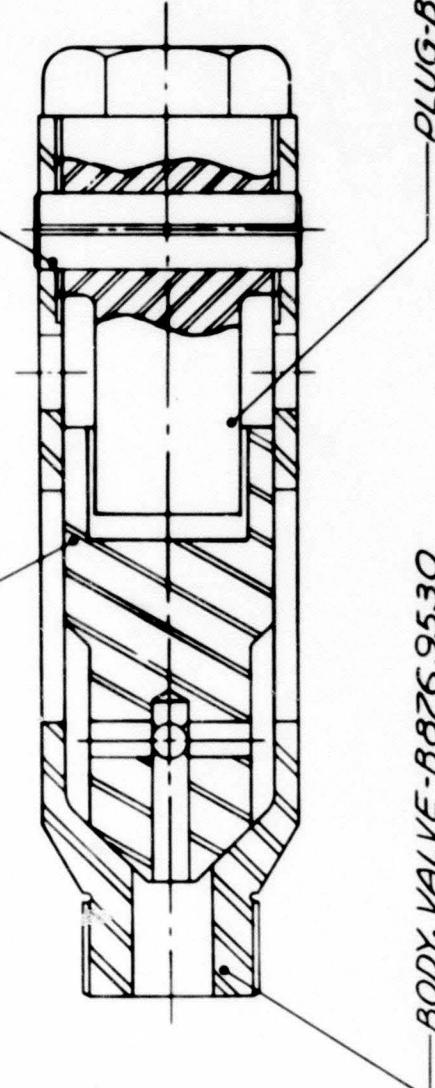


REVISIONS	DATE APPROVED

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PIN, SPRING MS 16362-12

VALVE-B 88769383



BODY, VALVE-B 88769530

PLUG-B 88769381

DEPT OF THE ARMY		WATERVILLE ARSENAL	
		WATERVILLE, N. Y.	
(RDI-72/9)		VALVE	
ASSEMBLY		ASSEMBLY	
SCALE	1/16	SCALE	1/16
UNIT BY		UNIT BY	
CASE		SHEET	
REMARKS		REMARKS	
ONE ENGINEERING RECORDS			
ARMY USE	ARMY USE	ARMY USE	ARMY USE
APPLICATION	GTV 9000	APPLICATION	GTV 9000
DO NOT APPLY PAINT TO	AS SPECIFIED	DO NOT	AS SPECIFIED
DO NOT	DO NOT	DO NOT	DO NOT

FIGURE 21.

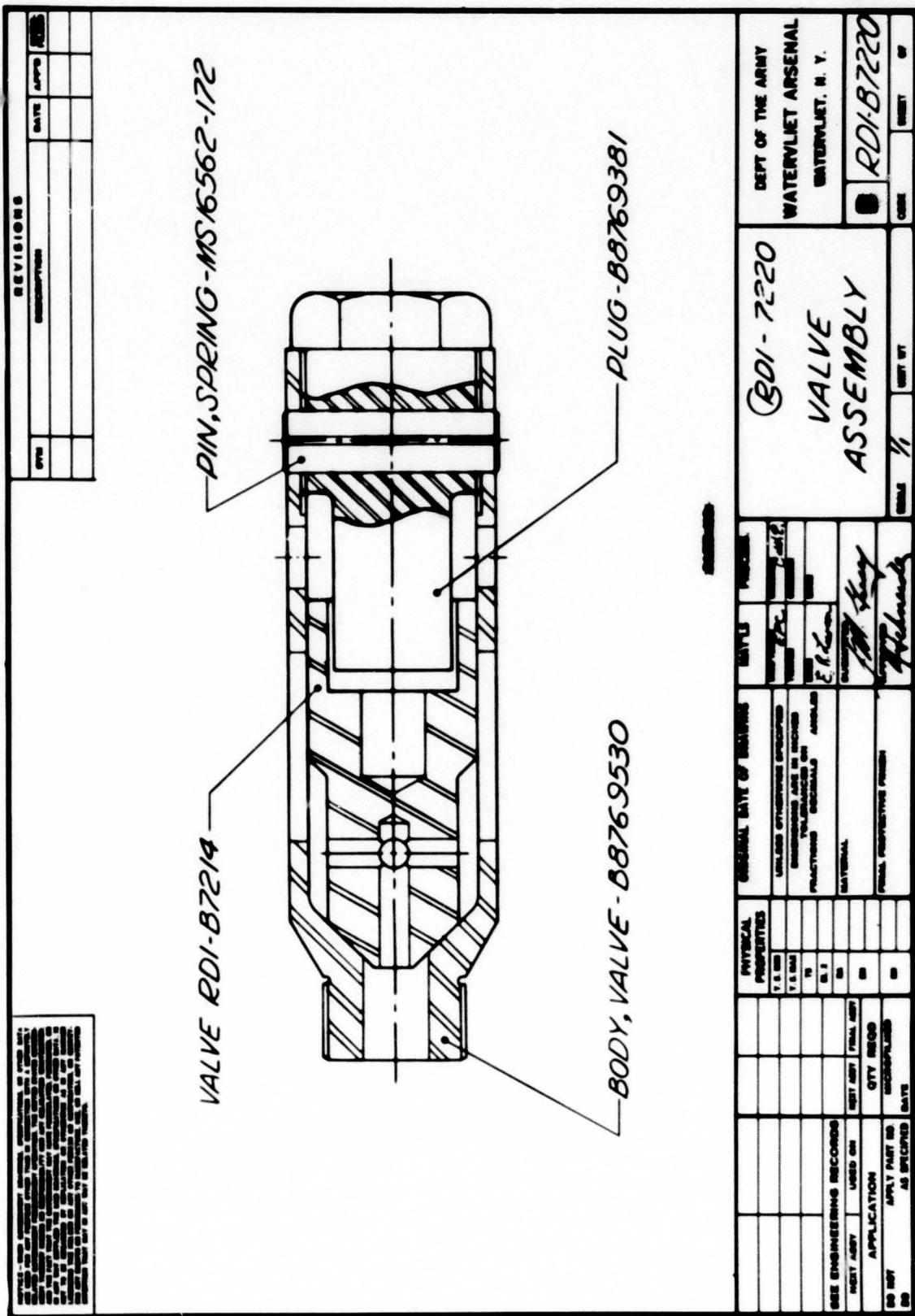


FIGURE 25

REVISIONS		DATE	APPROVED	FILED
REV.	DESCRIPTION			

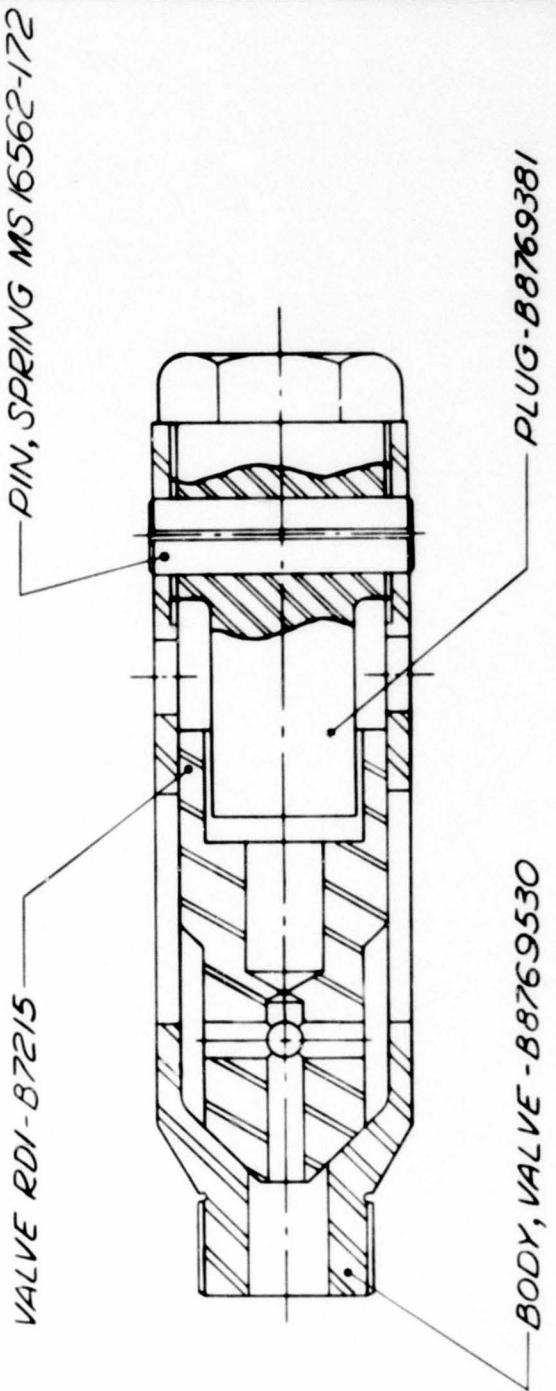
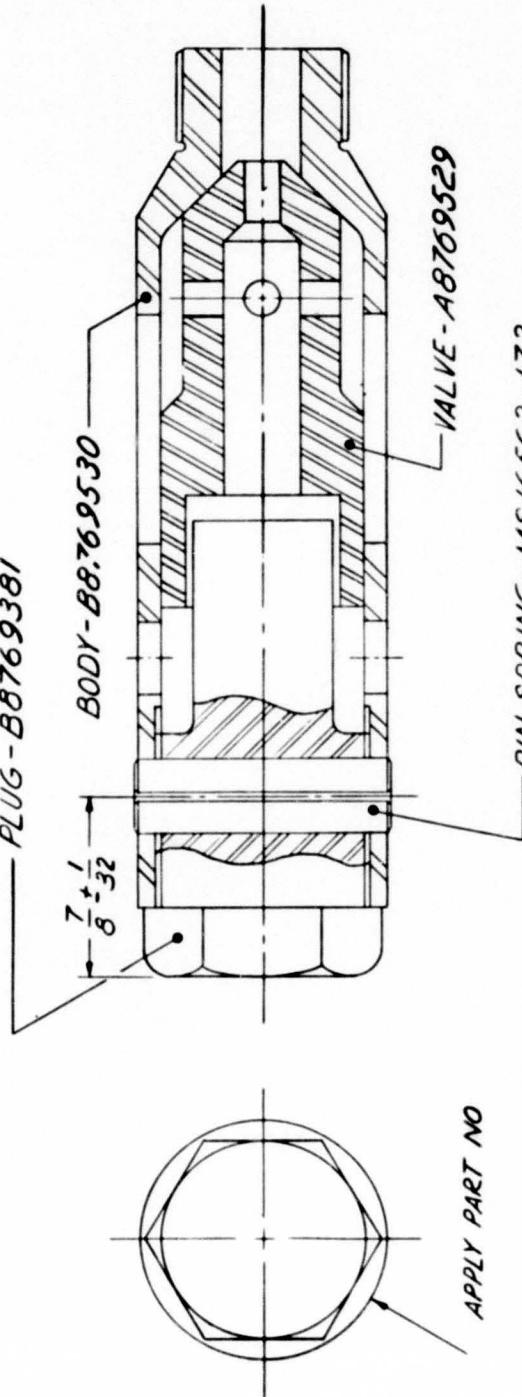


FIGURE 26

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REVISIONS		REV	DESCRIPTION	DATE	APPROV	FILED
0		NEW	SEE E.O. 13106			1/14

PLUG - B8769381



VALVE - A8769529

PIN, SPRING - MS16562-172  
(.375 DIA + .005 HOLE DRILL  
THROUGH AT ASSEMBLY)

PART NO. 8769531

FOR LIST OF PARTS SEE ENGINEERING PARTS LIST 8769531

PHYSICAL PROPERTIES	ORIGINAL DATE OF DRAWING DECEMBER 31, 1963	MATERIAL	PROCESS	DEPT OF THE ARMY	
				1 S. IRIN	1 S. IRAL
UNLESS OTHERWISE SPECIFIED	TRAMSMAN TRACER	WATERVLIET ARSENAL	CHEKED CRAZED		
DIMENSIONS ARE IN INCHES	TRACER	WATERVLIET, N. Y.	CHEKED		
FRACTIONS	TRACER	VALVE ASSEMBLY	CHEKED		
DECIMALS	TRACER		CHEKED		
ANGLES	TRACER		CHEKED		
MATERIAL	SUBMITTED		CHEKED		
FINAL PROTECTIVE FINISH	APPROVED		CHEKED		
APPLY PART NO AS SPECIFIED DATE	APPLIED		CHEKED		
REMARKS:	APPLIED		CHEKED		

FIGURE 27

REVISIONS	DESCRIPTION	SATE	APPRO	FILE

- BODY- B8769382

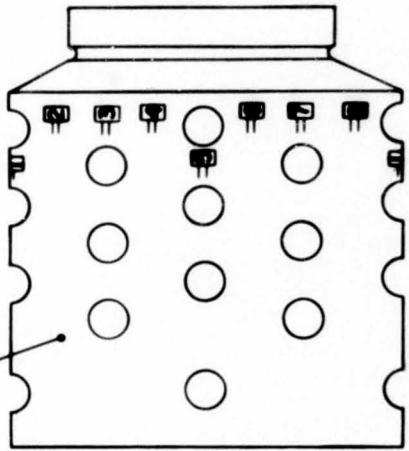
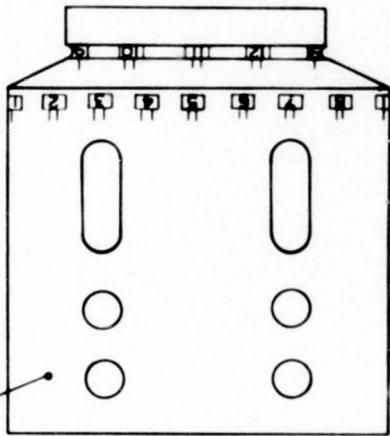


FIGURE 28

REVISIONS		DATE APPROVED																																																																		
<i>BODY- RDI-87227</i>																																																																				
<table border="1"> <thead> <tr> <th colspan="2">PHYSICAL PROPERTIES</th> <th>ORIGINAL DATE OF DRAWINGS</th> <th>REV'D</th> <th>APPROVED</th> </tr> </thead> <tbody> <tr> <td colspan="2"></td> <td>87</td> <td></td> <td></td> </tr> <tr> <td colspan="2">UNLESS OTHERWISE SPECIFIED</td> <td></td> <td></td> <td></td> </tr> <tr> <td colspan="2">DIMENSIONS ARE IN INCHES</td> <td></td> <td></td> <td></td> </tr> <tr> <td colspan="2">TOLERANCES ON</td> <td></td> <td></td> <td></td> </tr> <tr> <td colspan="2">FRACTIONAL DECIMALS</td> <td></td> <td></td> <td></td> </tr> <tr> <td colspan="2">ANGLED</td> <td></td> <td></td> <td></td> </tr> <tr> <td colspan="2">MATERIAL</td> <td></td> <td></td> <td></td> </tr> <tr> <td colspan="2">SEE ENGINEERING RECORDS</td> <td></td> <td></td> <td></td> </tr> <tr> <td>NEXT ACT</td> <td>USED ON</td> <td>LAST ACT</td> <td>FINAL ACT</td> <td></td> </tr> <tr> <td colspan="2">APPLICATION</td> <td>QTY REQD</td> <td>IN</td> <td></td> </tr> <tr> <td colspan="2">DO NOT APPLY PART NO.</td> <td>MICROFILMED</td> <td>IN</td> <td></td> </tr> <tr> <td colspan="2">AS SPECIFIED</td> <td>DATE</td> <td></td> <td></td> </tr> </tbody> </table>				PHYSICAL PROPERTIES		ORIGINAL DATE OF DRAWINGS	REV'D	APPROVED			87			UNLESS OTHERWISE SPECIFIED					DIMENSIONS ARE IN INCHES					TOLERANCES ON					FRACTIONAL DECIMALS					ANGLED					MATERIAL					SEE ENGINEERING RECORDS					NEXT ACT	USED ON	LAST ACT	FINAL ACT		APPLICATION		QTY REQD	IN		DO NOT APPLY PART NO.		MICROFILMED	IN		AS SPECIFIED		DATE		
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DO NOT APPLY PART NO.		MICROFILMED	IN																																																																	
AS SPECIFIED		DATE																																																																		
<b>DEPT OF THE ARMY</b> <b>WATERVLIET ARSENAL</b> <b>WATERVLIET, N. Y.</b>																																																																				
<b>STRAIN-GAGE LOCATIONS</b>																																																																				
<i>RDI-87227</i>																																																																				
SCALE	1	UNIT WT																																																																		
CORE		SHEET	OF																																																																	

— BODY-88769530



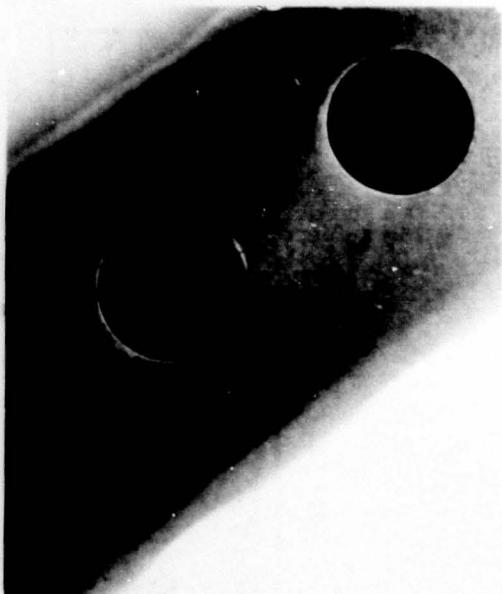
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		DEPT OF THE ARMY		WATERVILLE ARSENAL		WATERVILLE, N. Y.		RDI-B72228	
		<b>BODY DEVELOPMENT FOR STRAIN-GAGE LOCATIONS</b>							
ORIGINAL DATE OF DRAWING		MATERIAL		PROCESS				CODE SHEET OF	
PHYSICAL PROPERTIES		UNLESS OTHERWISE SPECIFIED		BT					
1/8 INCH		DIMENSIONS ARE IN INCHES		TACKER					
1/8 INCH		TOLERANCES ON		CUTTER					
1/16 INCH		FRACTIONS OR DECIMALS		ANGLED					
SEE ENGINEERING RECORDS		INCHES		CUT & TURNED					
NEXT ADRY		MATERIAL		SUBMITTED BY					
USED ON		NEXT ADRY FINAL ADRY		S. L. Tracy					
APPLICATION		QTY RECD		APPROVED					
DO NOT		APPLY PART NO		H. J. Hansen					
DO		AS SPECIFIED		DATE					

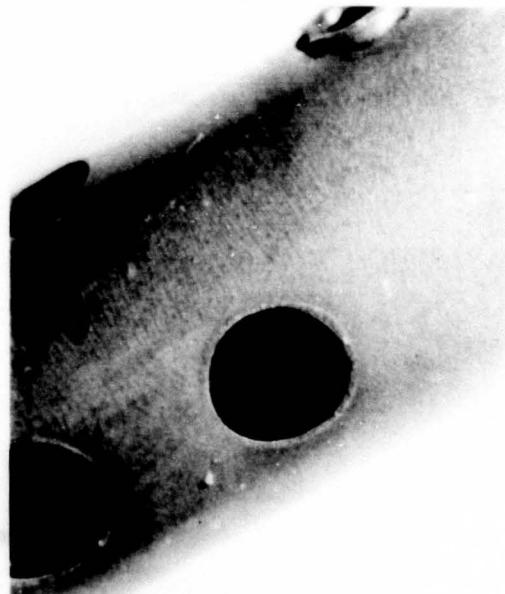
FIGURE 30

REVISIONS		DATE APPROVED	APPROVING OFFICER
REV N	DESCRIPTION		
<p><i>BODY - B 8769382</i></p>			
<p><b>BODY DEVELOPMENT FOR STRAIN-GAGE LOCATIONS</b></p>			
<b>DEPT OF THE ARMY</b> <b>WATERVILLE ARSENAL</b> <b>WATERVILLE, N. Y.</b>		<b>PDI-B7229</b>	
SCALE	1	UNIT WT	CORR. SHEET OF
<small>ORIGINAL DATE OF DRAWING</small> <b>MAY 1971</b> <small>PROCESS</small> <b>C-47</b> <small>UNLESS OTHERWISE SPECIFIED</small> <b>PRINT</b> <b>CHART</b> <small>DIMENSIONS ARE IN INCHES</small> <b>WEIGHT</b> <b>1000</b> <small>TOLERANCES ON</small> <b>SIZE</b> <b>1000</b> <small>FRACTIONS</small> <b>EL. 1</b> <small>DECIMALS</small> <b>E. I. L. C.</b> <small>AMERICAN</small> <small>MATERIAL</small> <small>SUPERVISOR</small> <i>[Signature]</i> <small>FINAL PROTECTIVE FINISH</small> <i>[Signature]</i> <small>INITIALS</small> <i>[Signature]</i>			
<b>BASE ENGINEERING RECORDS</b> <small>NEXT ARMY</small> <b>USED ON</b> <b>ARMY</b> <b>ARMY</b> <small>APPLICATION</small> <b>QTY REQD</b> <b>ARMY</b> <small>DO NOT APPLY PART NO.</small> <b>MICROFILMED</b> <b>ARMY</b> <small>AS SPECIFIED</small> <b>DATE</b> <b>ARMY</b>			

FIGURE 31



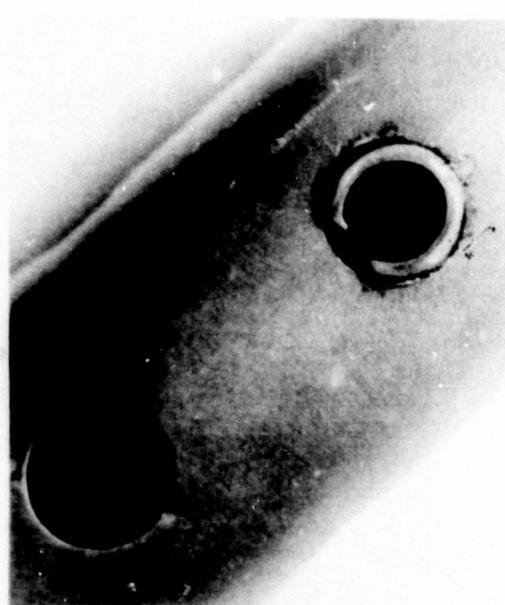
A



B



C



D

Figure 32: Four Views of the Valve Body to which "Stress Coat" had been applied.

A and B are in Line.  
C and D are 90° from A and B and in line.



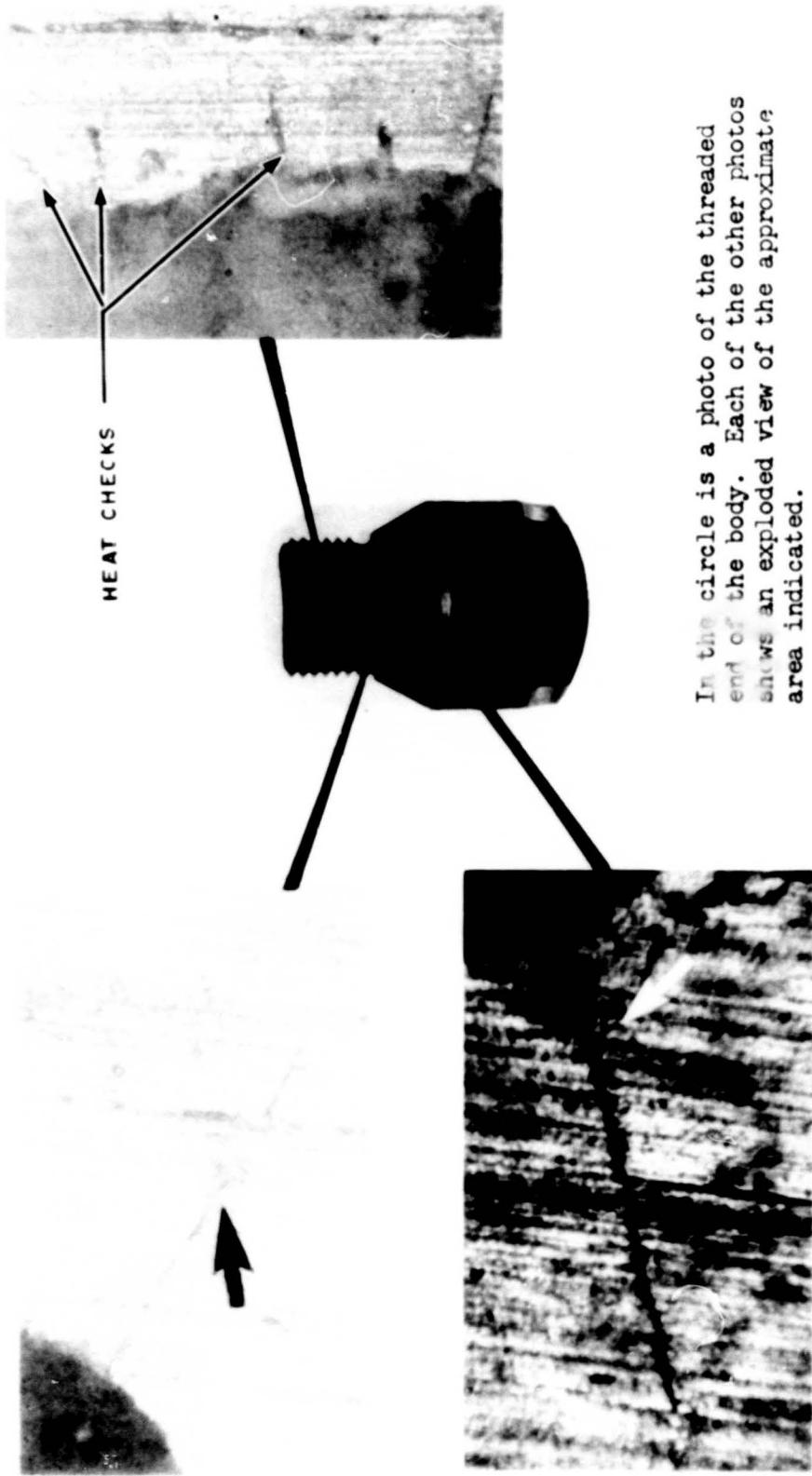
**Figure 33: View showing Magnetic Particle Indication  
On S.V. 1 after 1000 rounds.**

**Arrow Points out Indication**



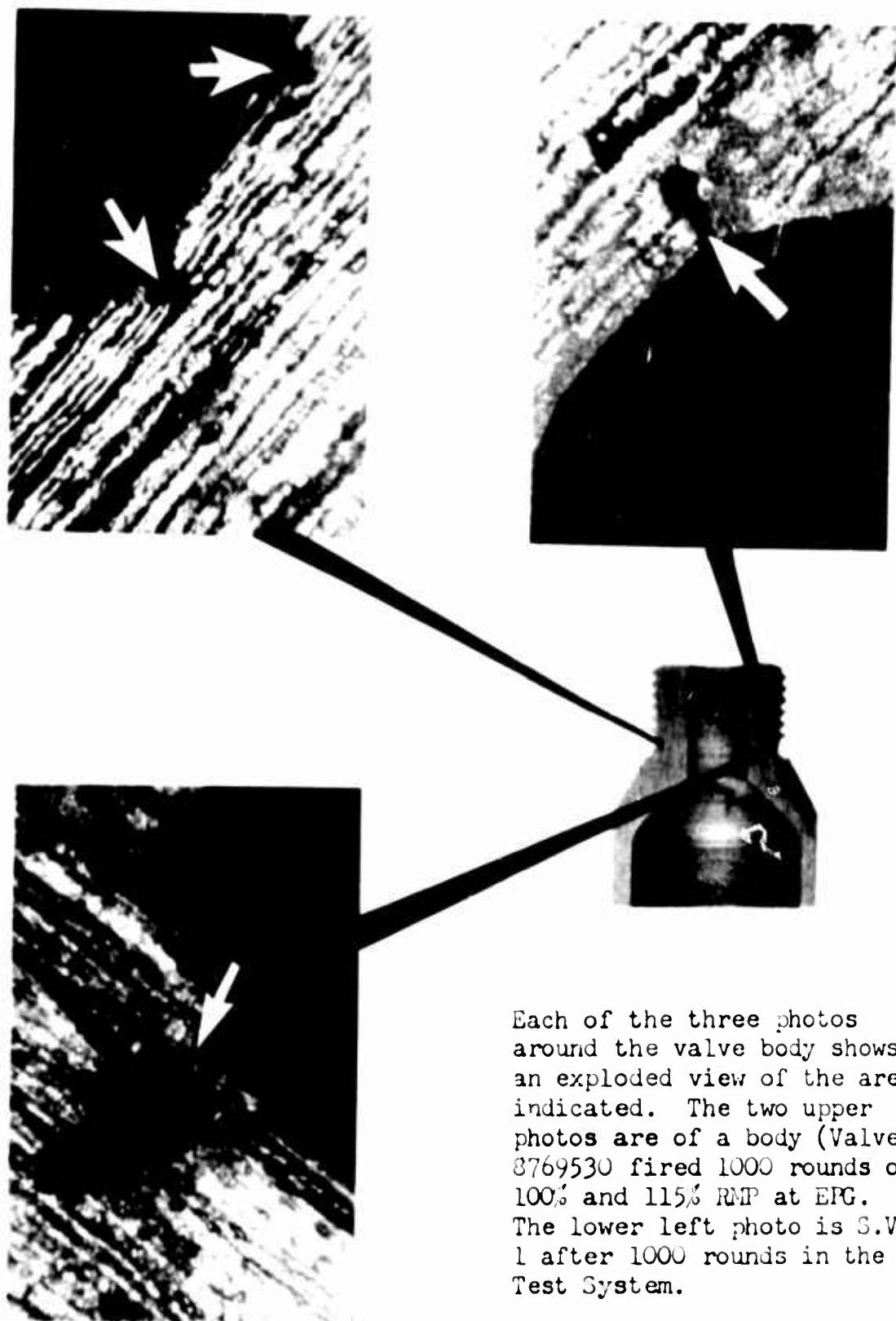
Figure 34: View showing Magnetic Particle Indication  
on M.V. 3 after 10,000 Rounds.

Arrow Points out Indication



In the circle is a photo of the threaded end of the body. Each of the other photos shows an exploded view of the approximate area indicated.

Figure 35: Photomicrographs of M.V. 3 after 10,000 rounds;  
Macroetched with 50% HCl Solution and Magnified 65x



Each of the three photos around the valve body shows an exploded view of the area indicated. The two upper photos are of a body (Valve) 8769530 fired 1000 rounds of 100% and 115% RMP at EPG. The lower left photo is S.V. 1 after 1000 rounds in the Test System.

Figure 36: Photomacrographs Of Valve Bodies Macroetched with 50% HCL Solution and Magnified 65x

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AD Watervliet Arsenal, Watervliet, N. Y.	Accession No.	AD Watervliet Arsenal, Watervliet, N. Y.	Accession No.
UNCLASSIFIED Artillery Howitzers Cannon, 155mm T1255E3	BORE EVACUATOR VALVE TEST, CANNON 155MM HOWITZER, M126 by J. M. Gieseley, Mechanical Engineer; E. R. Lawson, Mechanical Engineer and R. L. Rosenblum, Mechanical Engineer.  Report No. WWT-11-6412, August 1964, 60 pages; 4 tables and 36 figures. OQMS Code No. 4020.24.2223.2.10.04 Unclassified Report	Bore Evacuation Valves Bore Scavenging  The limited life of Bore Evacuator Valve Assembly 8769384 during firing tests led to the authorization of a test program to find a valve assembly with a longer life. The cost of testing in the gun (155mm How. M126) made it economical to build a test apparatus which simulated the weapon. The test program was the basis for the incorporation of valve assembly 8769531 into the weapon system. A comparison of the strain level of the modification is presented. The life of the then current production valve assembly and the new production valve assembly under different charges is also given.	Bore Evacuation Valves Bore Scavenging  The limited life of Bore Evacuator Valve Assembly 8769384 during firing tests led to the authorization of a test program to find a valve assembly with a longer life. The cost of testing in the gun (155mm How. M126) made it economical to build a test apparatus which simulated the weapon. The test program was the basis for the incorporation of valve assembly 8769531 into the weapon system. A comparison of the strain level of the modification is presented. The life of the then current production valve assembly and the new production valve assembly under different charges is also given.
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